

AWT AERODYNAMIC DESIGN STATUS

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SVERDRUP CORPORATION

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ALTITUDE WIND TUNNEL
FOR PROPULSION AND ICING RESEARCH

NASA LEWIS RESEARCH CENTER, CLEVELAND, OHIO

PRELIMINARY ENGINEERING REPORT
SECTION 2

TUNNEL AERODYNAMICS, PERFORMANCE
AND OPERATING COST

PREPARED BY
SVERDRUP CORPORATION
St. Louis, Missouri

FOR
NATIONAL AERONAUTICS AND SPACE ADMINISTRATION
LEWIS RESEARCH CENTER
CLEVELAND, OHIO
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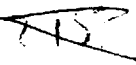
Barbara Pennington

Bill Crouch, et al

25-PERCENT REVIEW, AWT AERODYNAMICS

MAIN ACTIVITIES

- ANALYSIS OF AN INDEPENDENT PLENUM EVACUATION SYSTEM (PES)
- AIRLINE DEFINITION AND PRESSURE LOSS CODE DEVELOPMENT WITH EMPHASIS ON THE DEVELOPMENT OF ALTERNATE COOLER AND BASELINE SCAVENGING SCOOP DESIGNS
- CONTRACTION GEOMETRY AND CODE ANALYSIS
- AERODYNAMIC DESIGN OF THE TWO-STAGE FAN USING THE REF. 1 DEFINED PRESSURE RATIO REQUIREMENTS
- COORDINATION AND COMMUNICATION



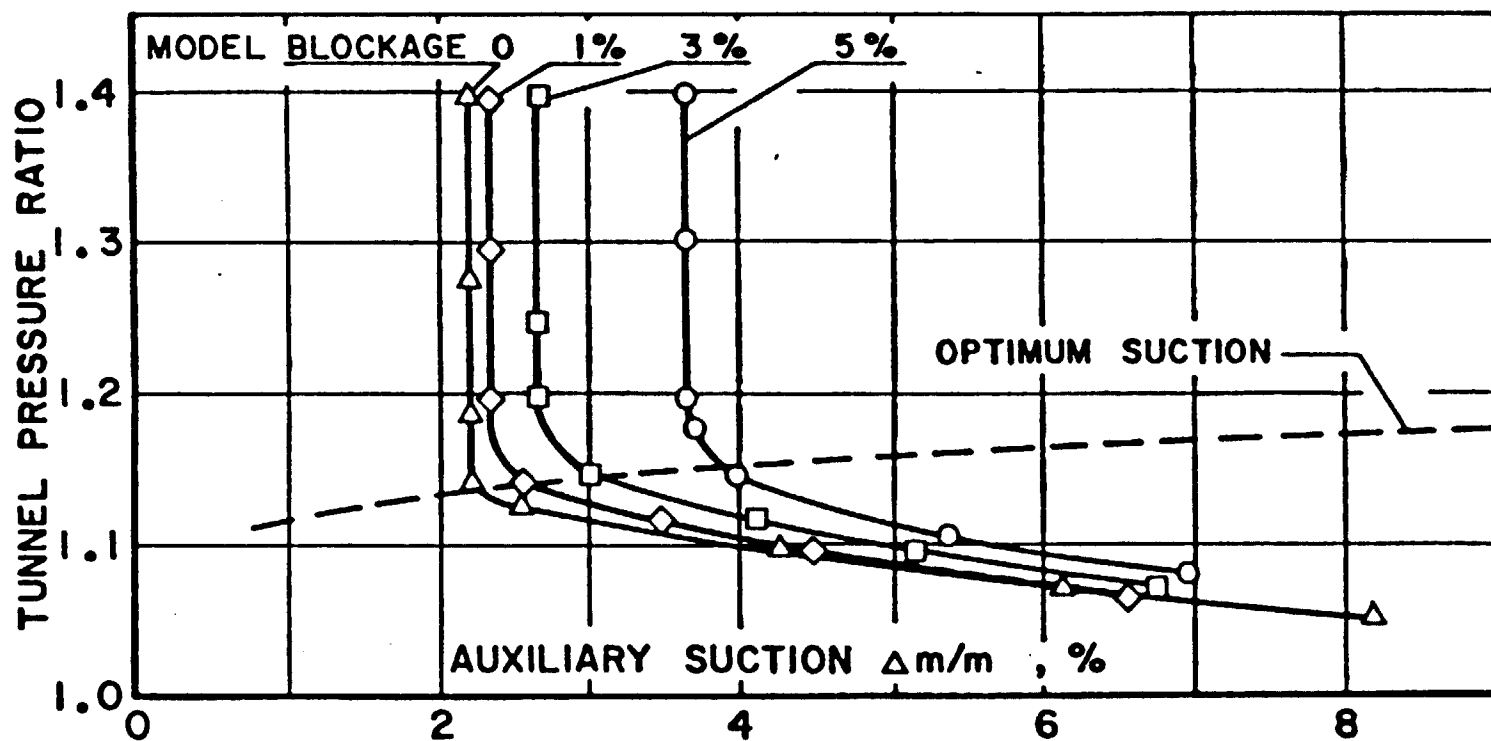
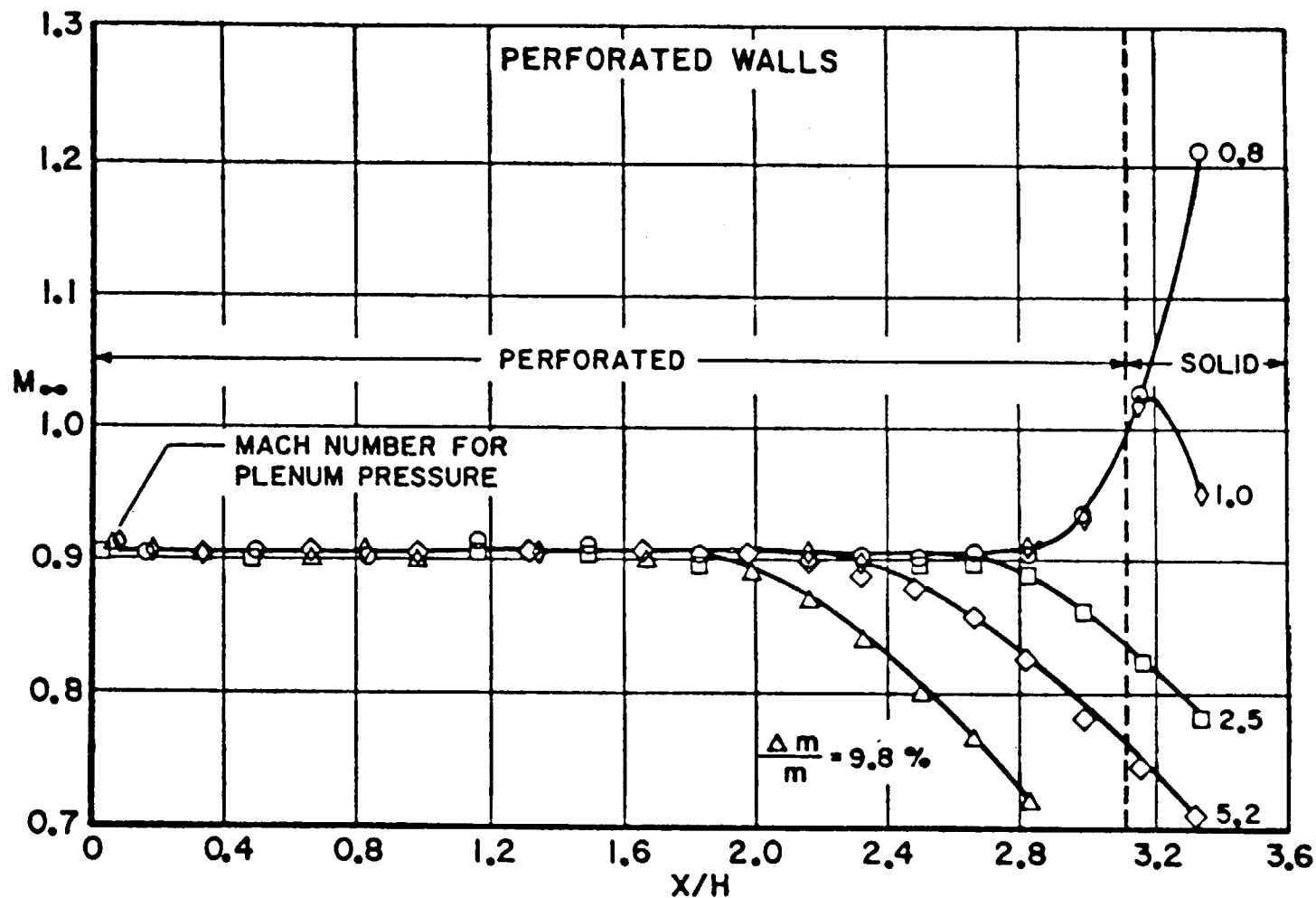


FIG. 14.13. Influence of model size on pressure ratio and suction requirements of perforated wind tunnel at $M = 1.0$ (22.5 per cent open walls, $\frac{1}{4}$ in. holes, parallel walls) (AEDC 1 ft transonic model tunnel⁴).

MACH NUMBER DISTRIBUTIONS FOR PERFORATED TEST SECTION WITH PARALLEL WALLS FOR VARIOUS PLENUM CHAMBER SUCTION QUANTITIES



Sverdrup

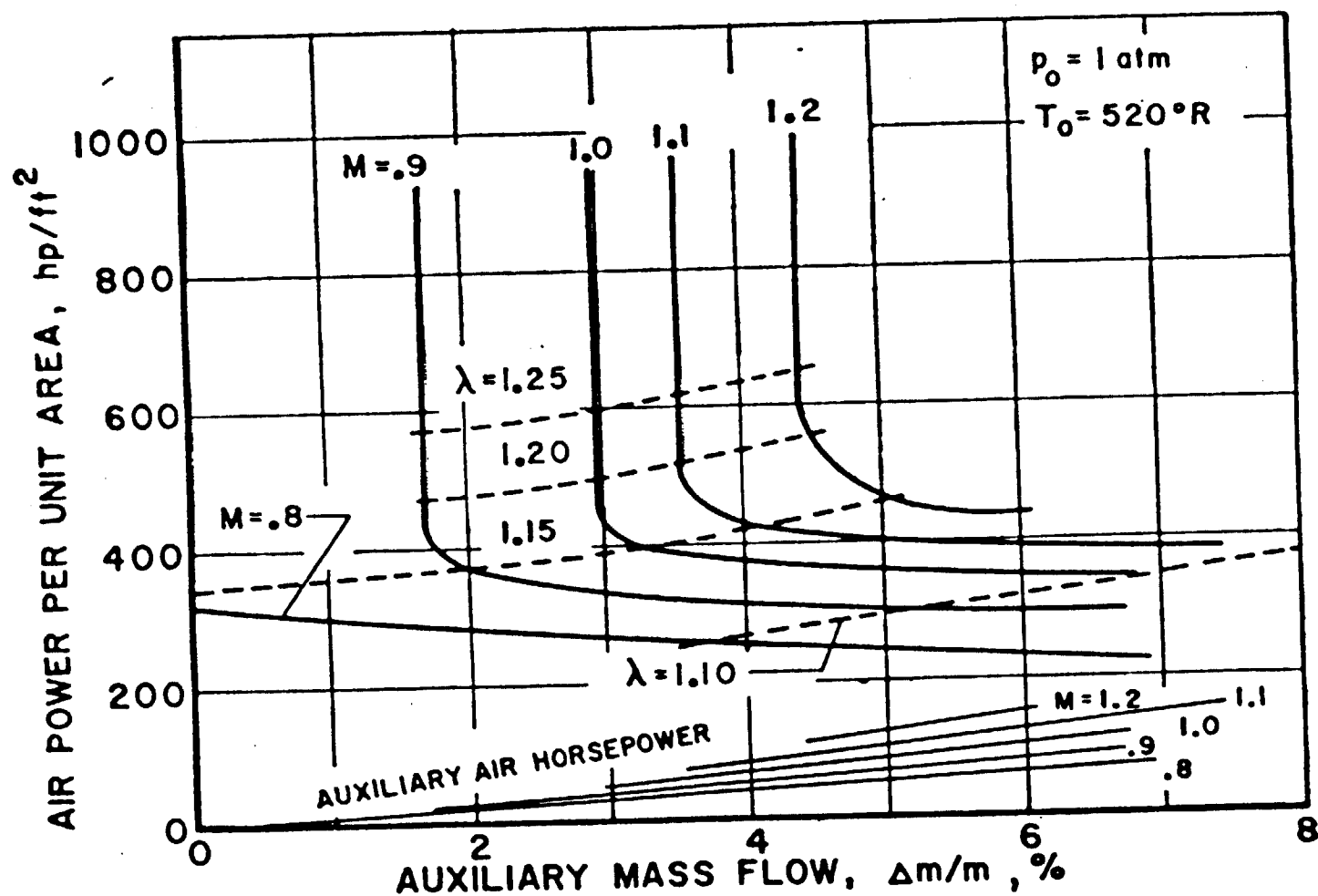


FIG. 14.16. Power requirements and auxiliary mass flow of slotted test section for various Mach numbers (sixteen slots, 11 per cent open, diffuser flaps closed) (AEDC 1 ft transonic model tunnel⁵).

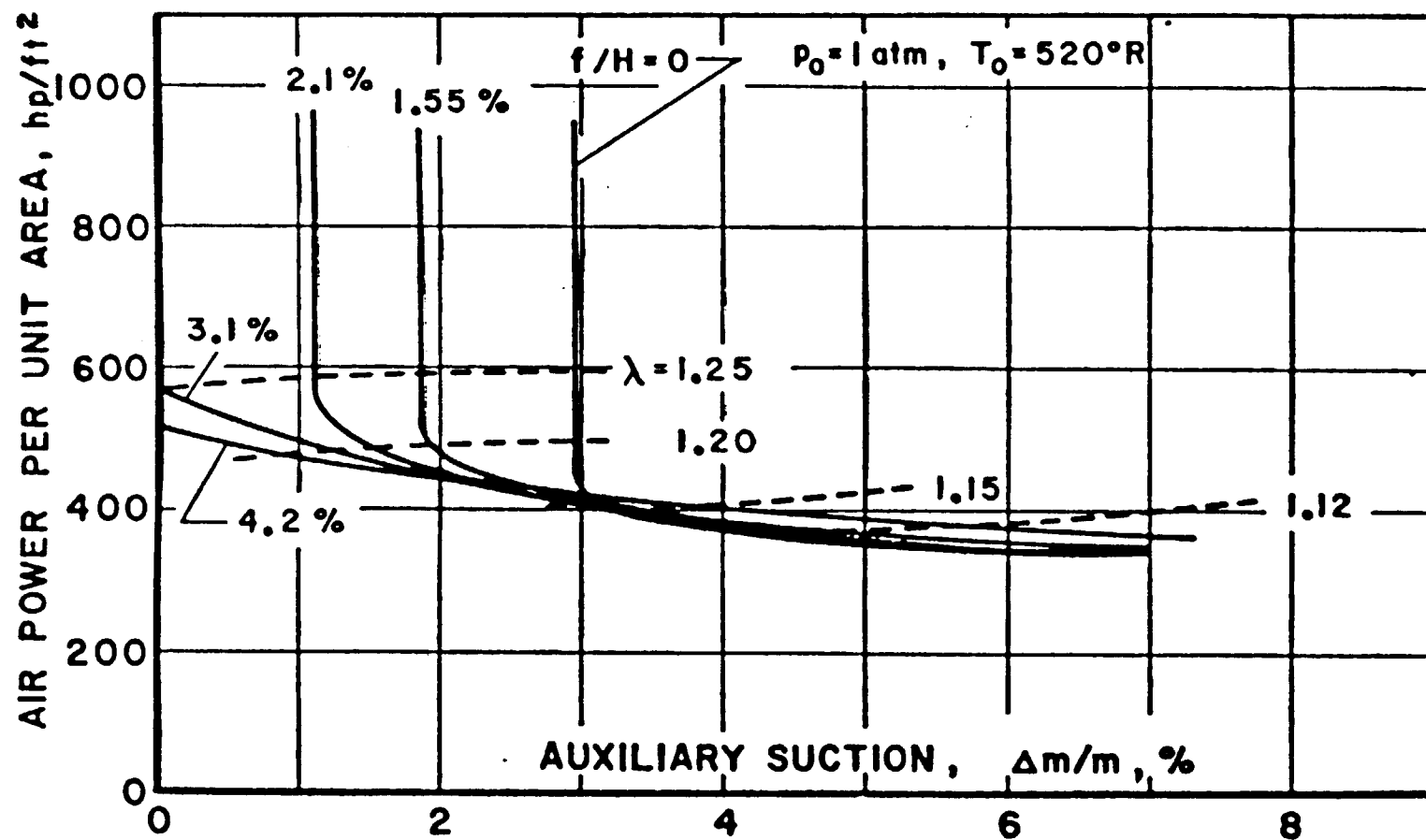


FIG. 14.17. Influence of diffuser flap opening on power requirements of slotted test section at $M = 1.0$ (sixteen slots; 11 per cent open, parallel walls) (AEDC 1 ft transonic model tunnel⁵).

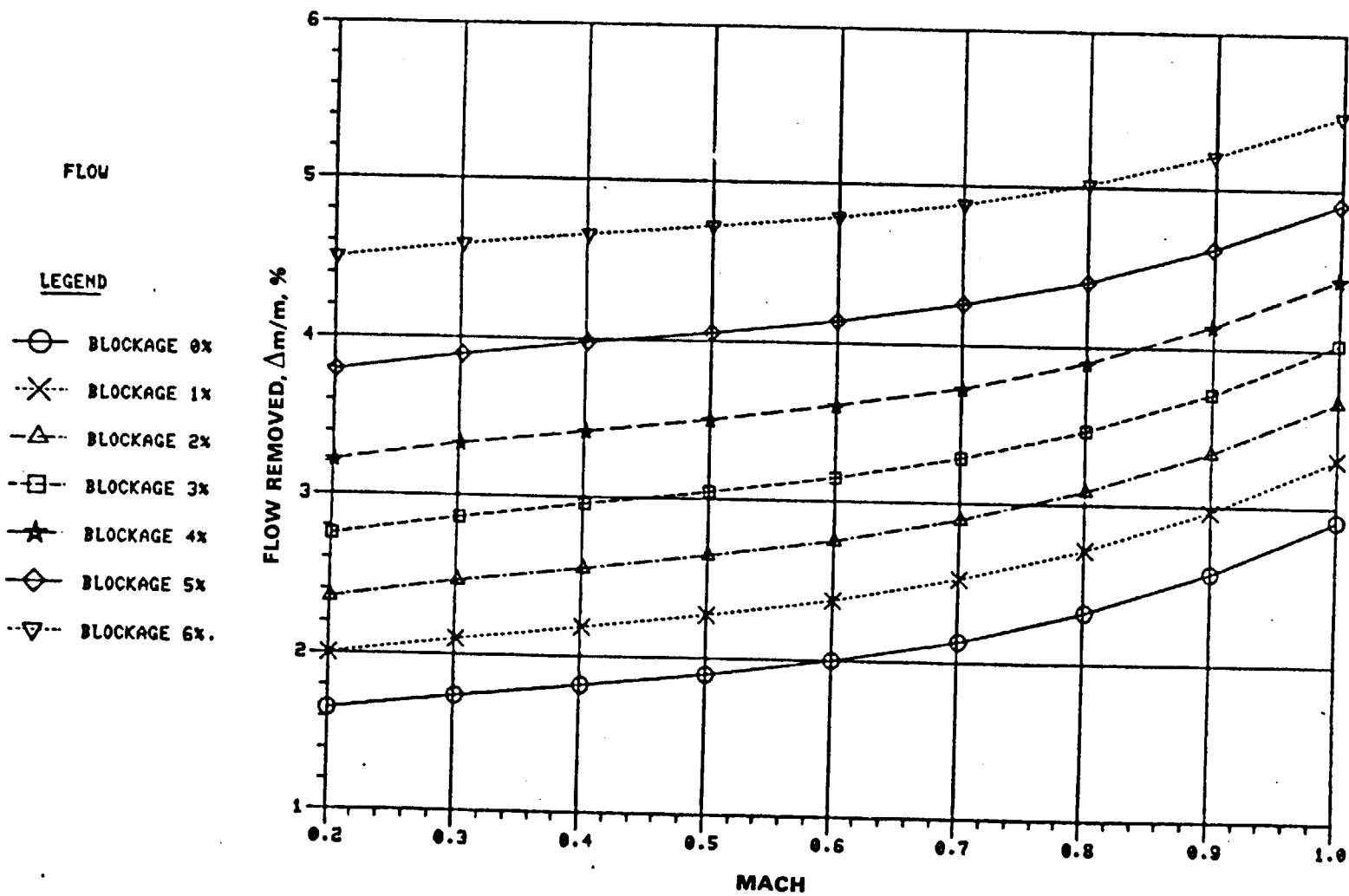


FIGURE 2. PLENUM FLOW REMOVAL REQUIREMENTS FOR AWT

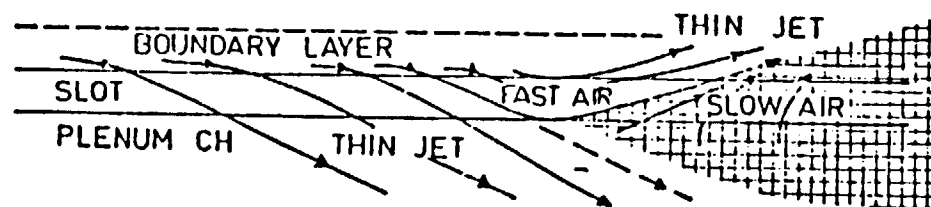
THE FIRST DIFFUSER ALGORITHM

Baseline Tunnel

$$B_t = B_{t,o} + \left(\frac{\Delta m}{m} \right), z$$

PES Tunnel

$$B_t = B_{t,o}, z$$

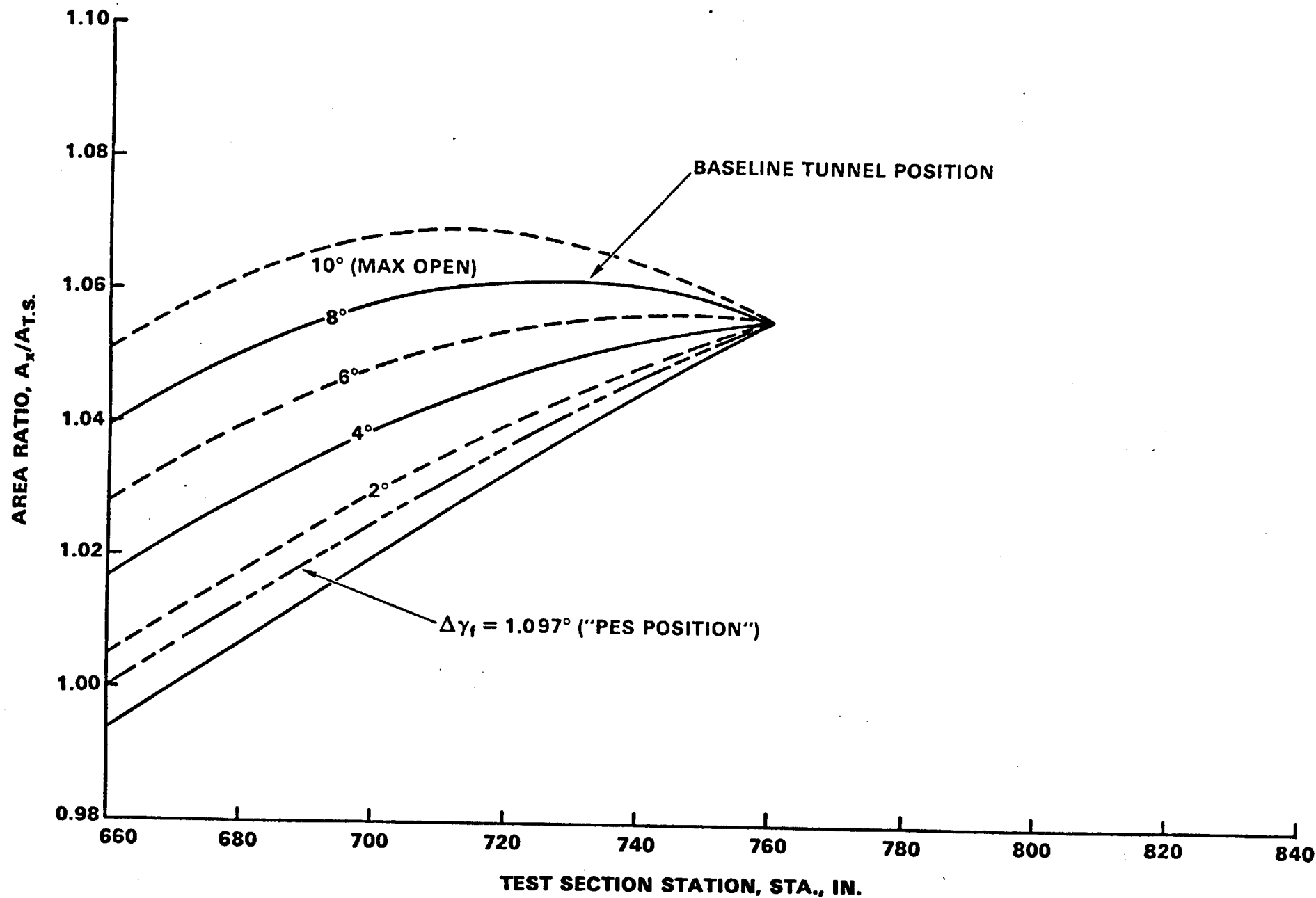


NYBERG'S SLOT FLOW MODEL

$$B_{t,o} = \left[1 - \frac{[A_{660}]e}{45,726.4} \right] 100, z$$

$$[A_{660}]e = (233.68 - 2\delta_{ex}^*)^2 (0.429705) + (236.24 - 2\delta_e^*)^2 (0.39889), \text{ in}^2$$

$$\delta_{ex}^* = 10.38275 \left[\frac{R_e}{t} \right]^{-1/5}, \text{ in}$$



**FIGURE 1. EFFECT OF EJECTOR FLAP OPENING ON
DIFFUSER GEOMETRIC AREA DISTRIBUTION**

CROSS-LEG DIFFUSER ALGORITHM & TEST LEG PRESSURE RATIO

- VARNER'S CODE FOR FIRST DIFFUSER DEFINES DIFFUSER EXIT BOUNDARY-LAYER BLOCKAGE
- THIS BLOCKAGE IS USED AS AN ARGUMENT FOR A CROSS-LEG DIFFUSER CALCULATION FURTHER ACCENTUATING THE INFLUENCE OF AN INDEPENDENT PES
- FOR CALCULATION AROUND THE MATCHED-ALTITUDE ENVELOPE (FIG. 5), THE PES-TUNNEL IS ALSO GIVEN THE ADVANTAGE THAT THE $\Delta m/m$ MASS FLOW DEFICIT REDUCES THE DIFFUSER ENTRANCE AND CORNER 1 MACH NUMBER (FIG. 6)
- THE RESULTING REDUCTION IN THE TEST LEG PRESSURE RATIO IS SHOWN IN (FIG. 7)

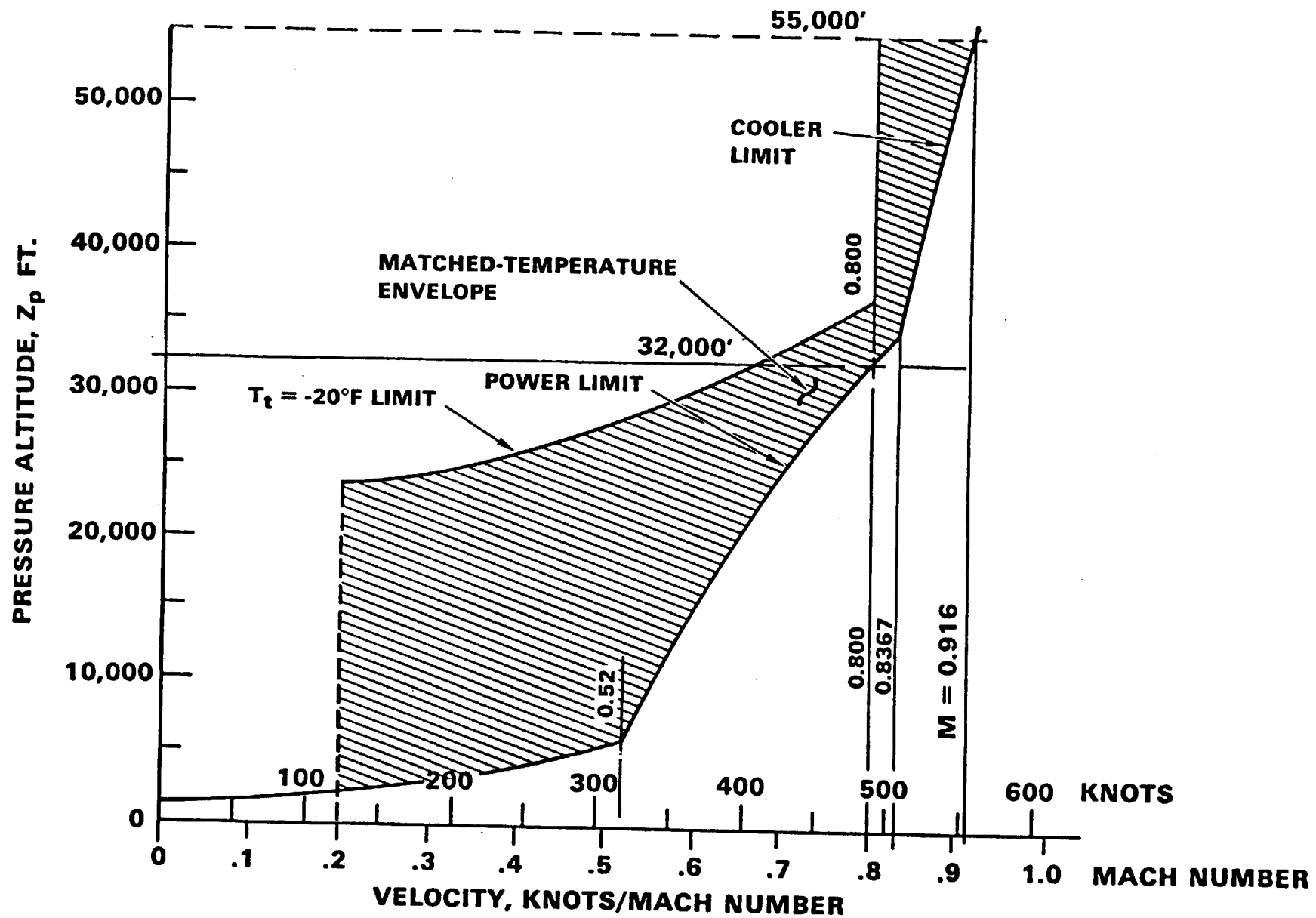


FIGURE 5. AWT PERFORMANCE 20' TEST SECTION

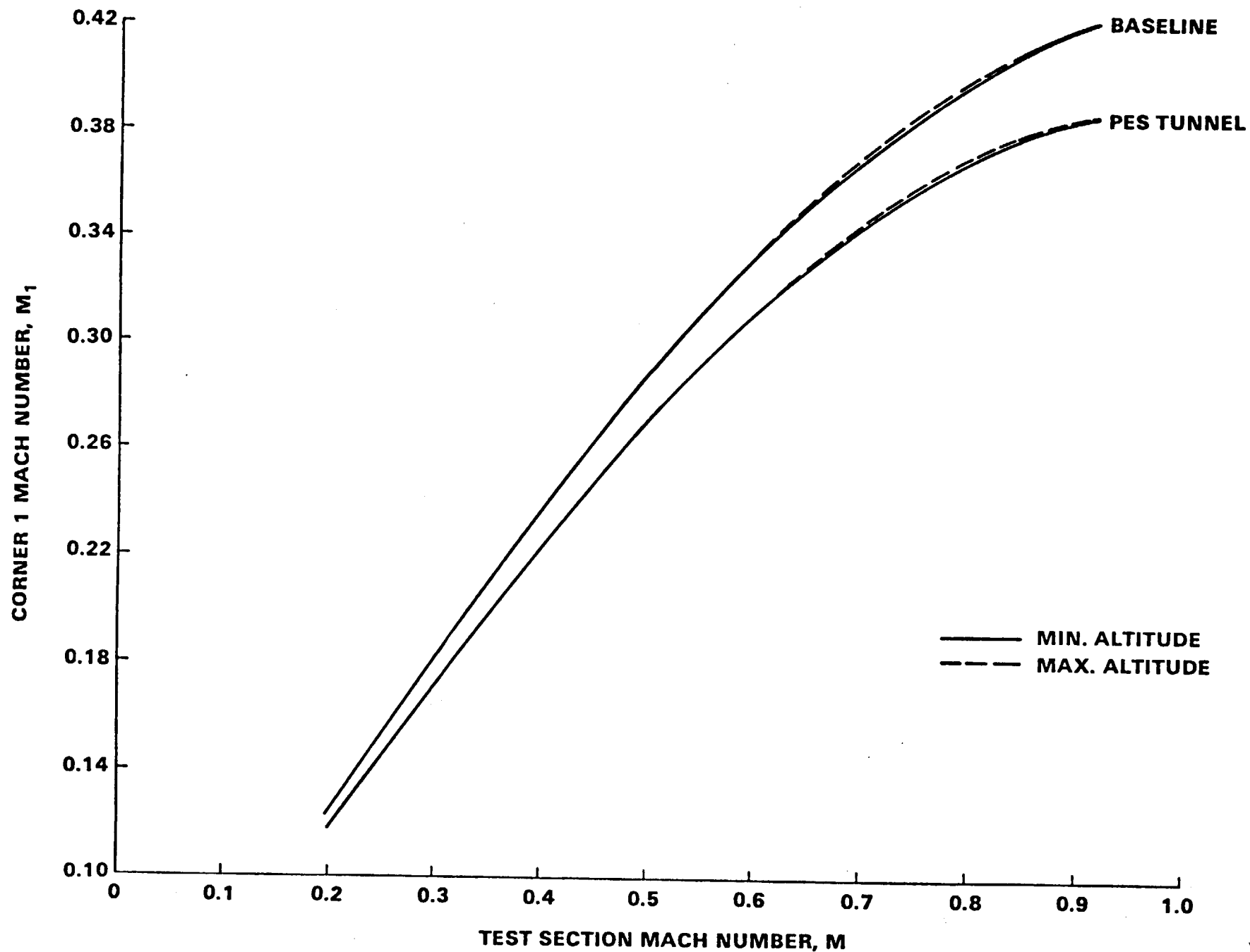


FIGURE 6. EFFECT OF PES SCHEME ON FIRST CORNER ENTRANCE MACH NUMBER, 6% MODEL CW

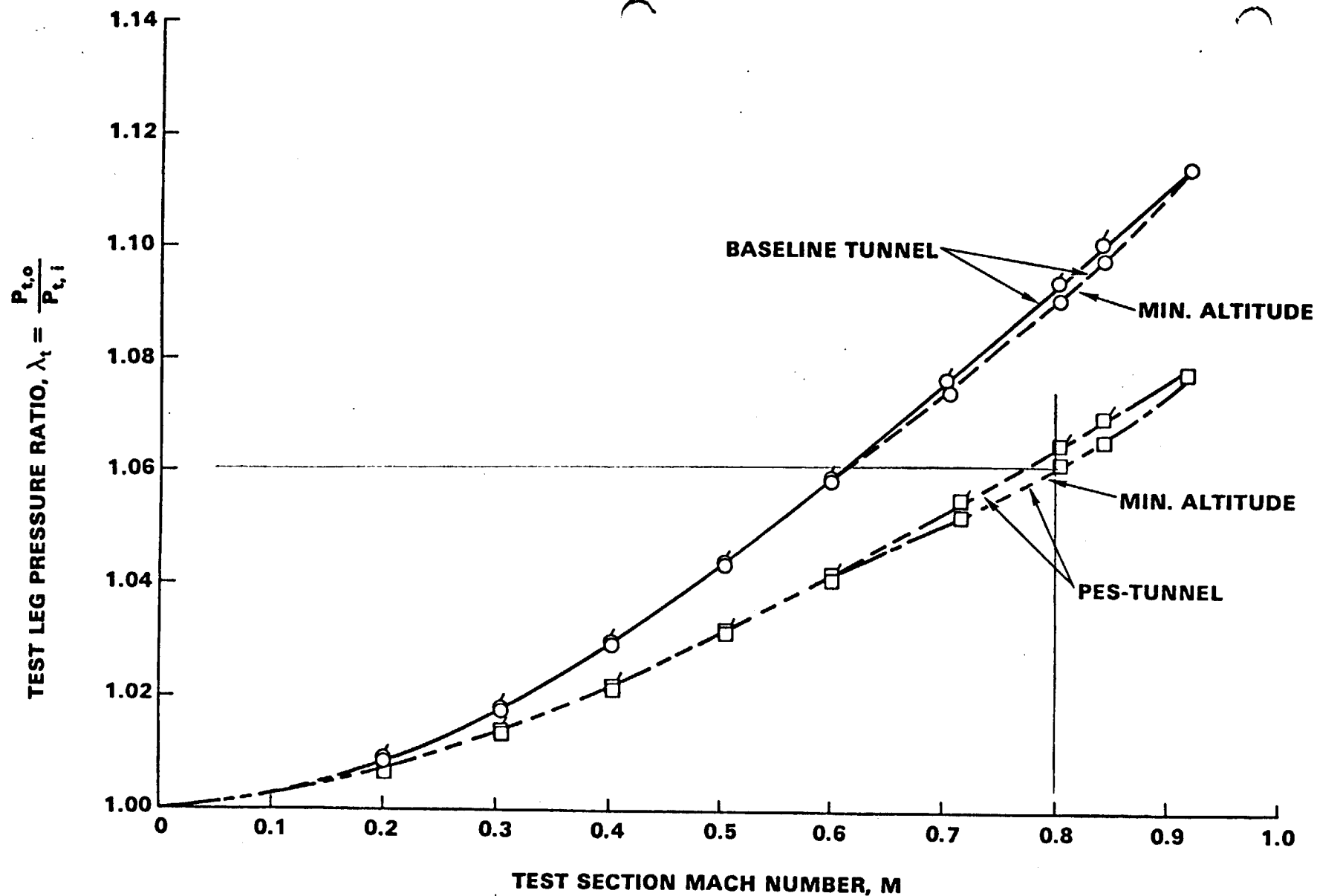
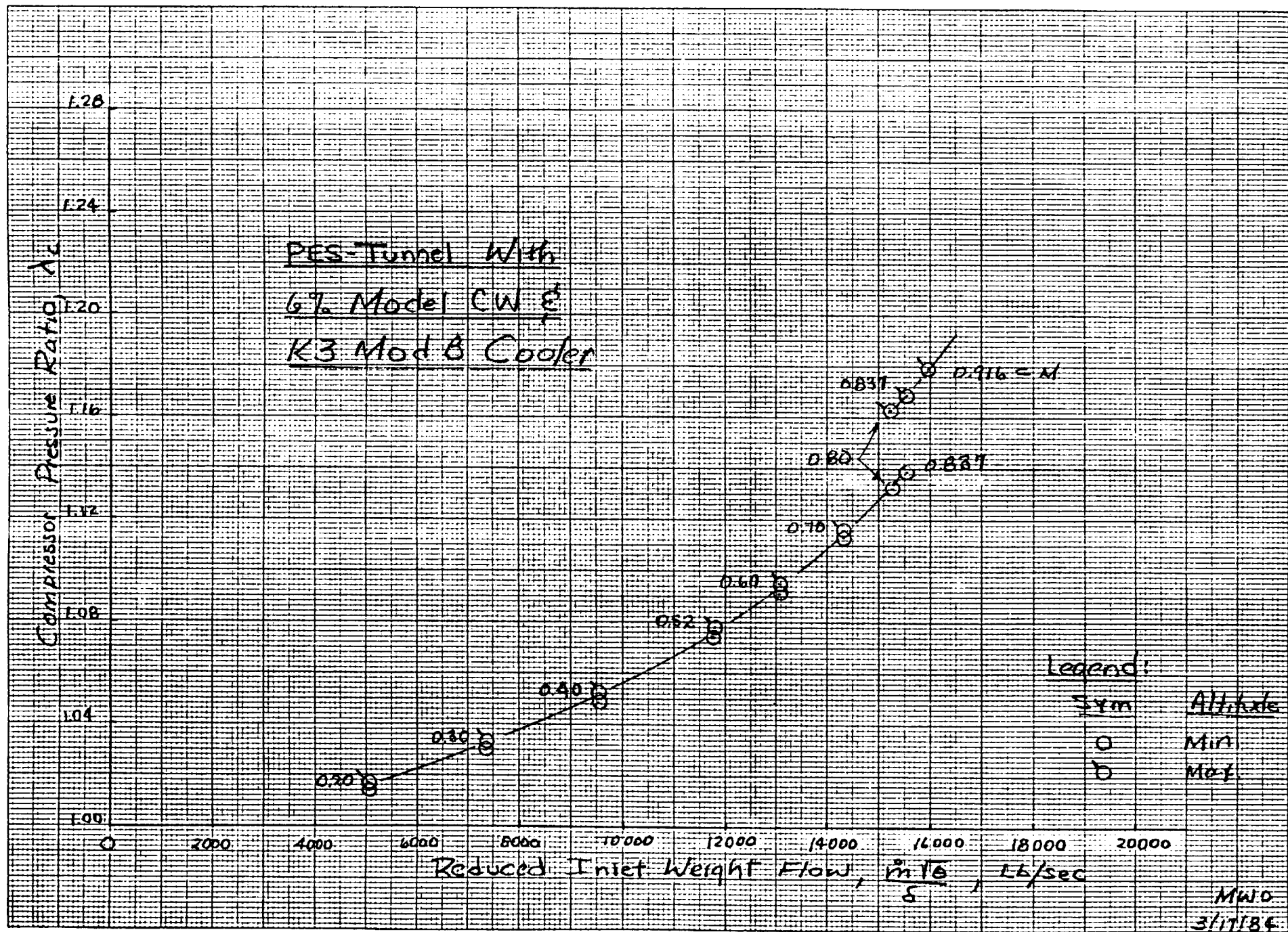
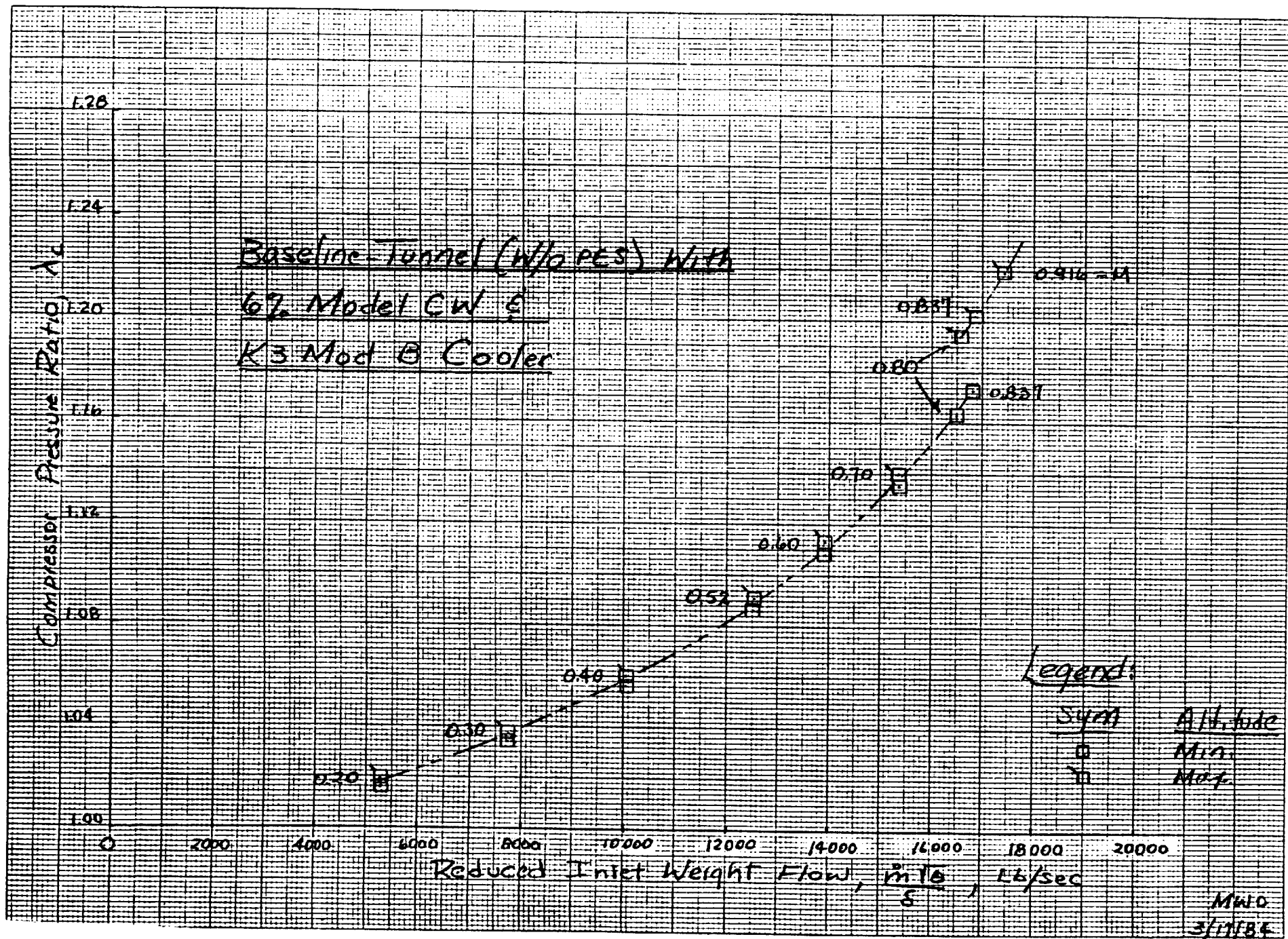
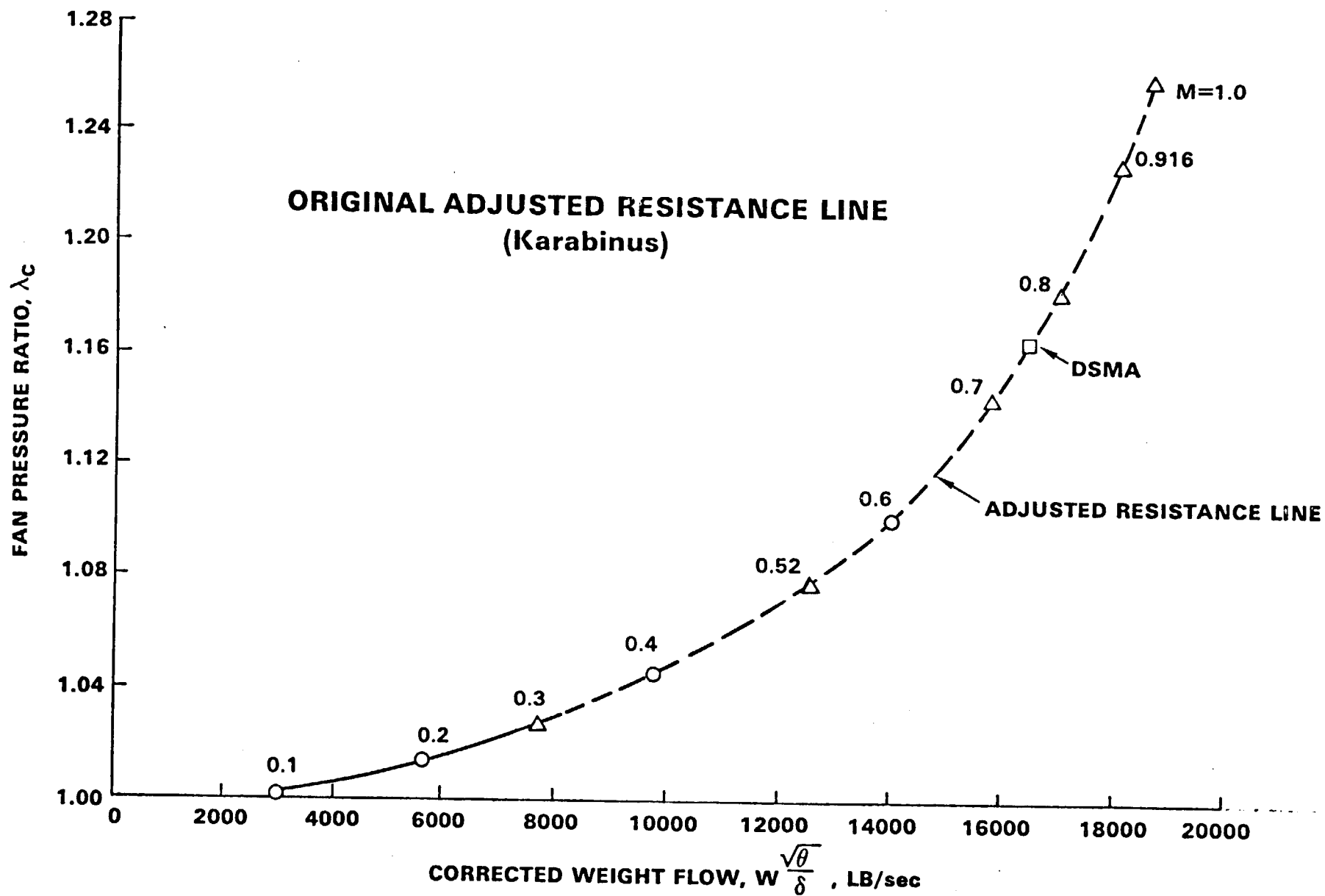
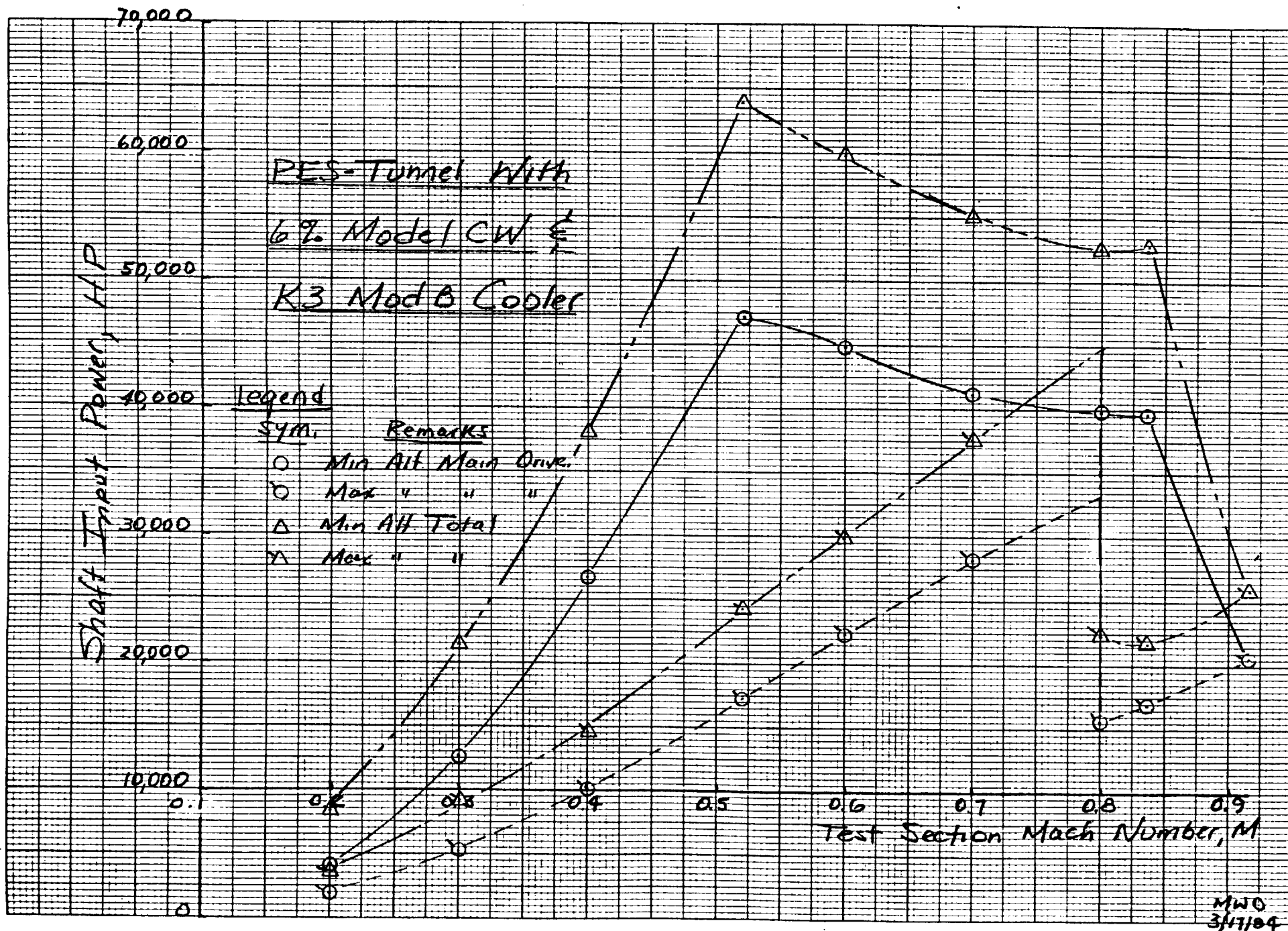


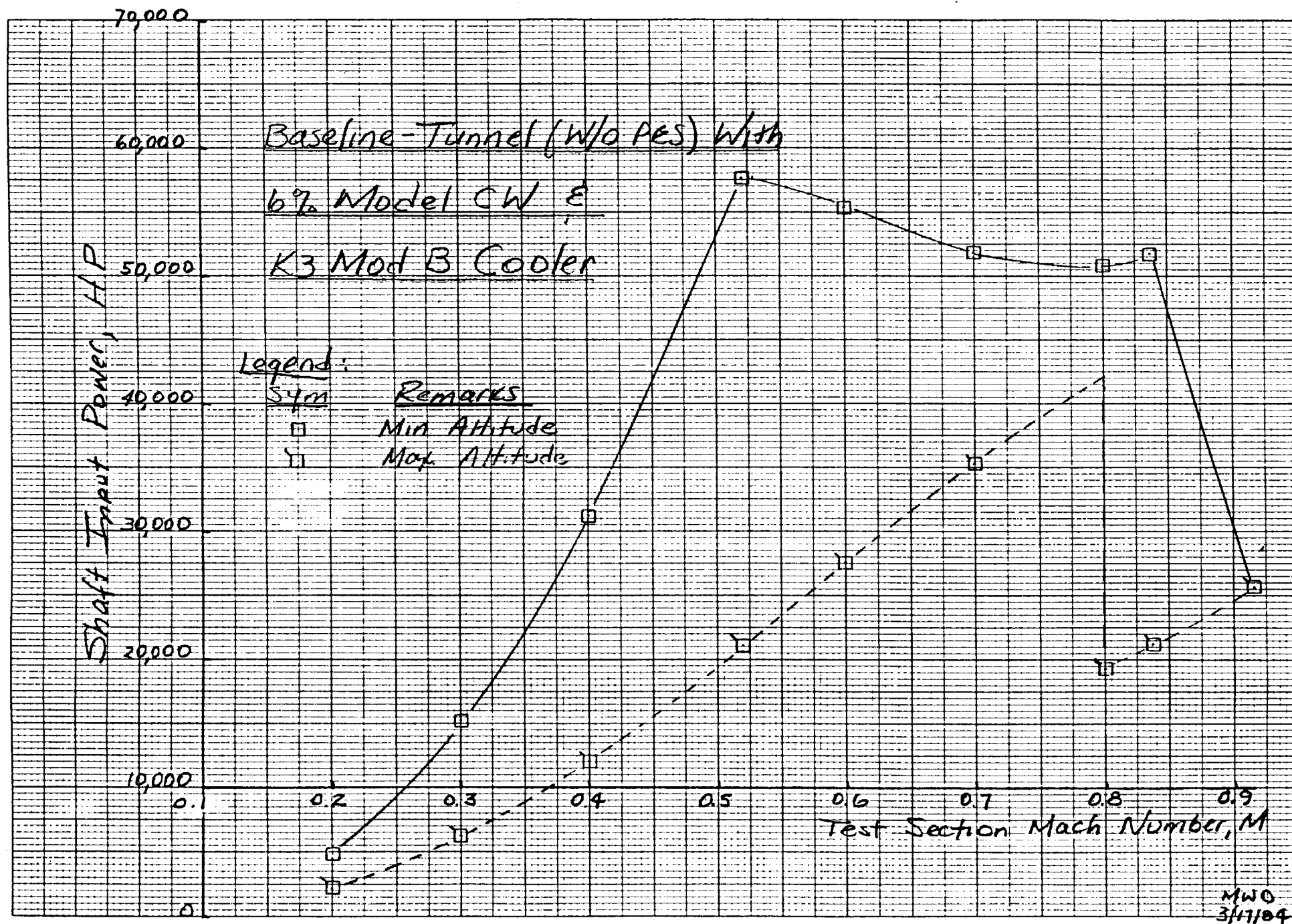
FIGURE 7. EFFECT OF PES SCHEME ON TEST LEG PRESSURE RATIO WITH 6% MODEL CW











Adiabatic Efficiency, η_c

1.00

0.90

0.80

0.70

0.6

0.1

0.2

0.3

0.4

0.5

0.6

0.7

0.8

0.9

Test Section Mach Number, M

Legend

Sym

○

○

□

□

Remarks

Min Alt. PES-Tunnel

Max Alt. PES-Tunnel

Min Alt. W/o PES

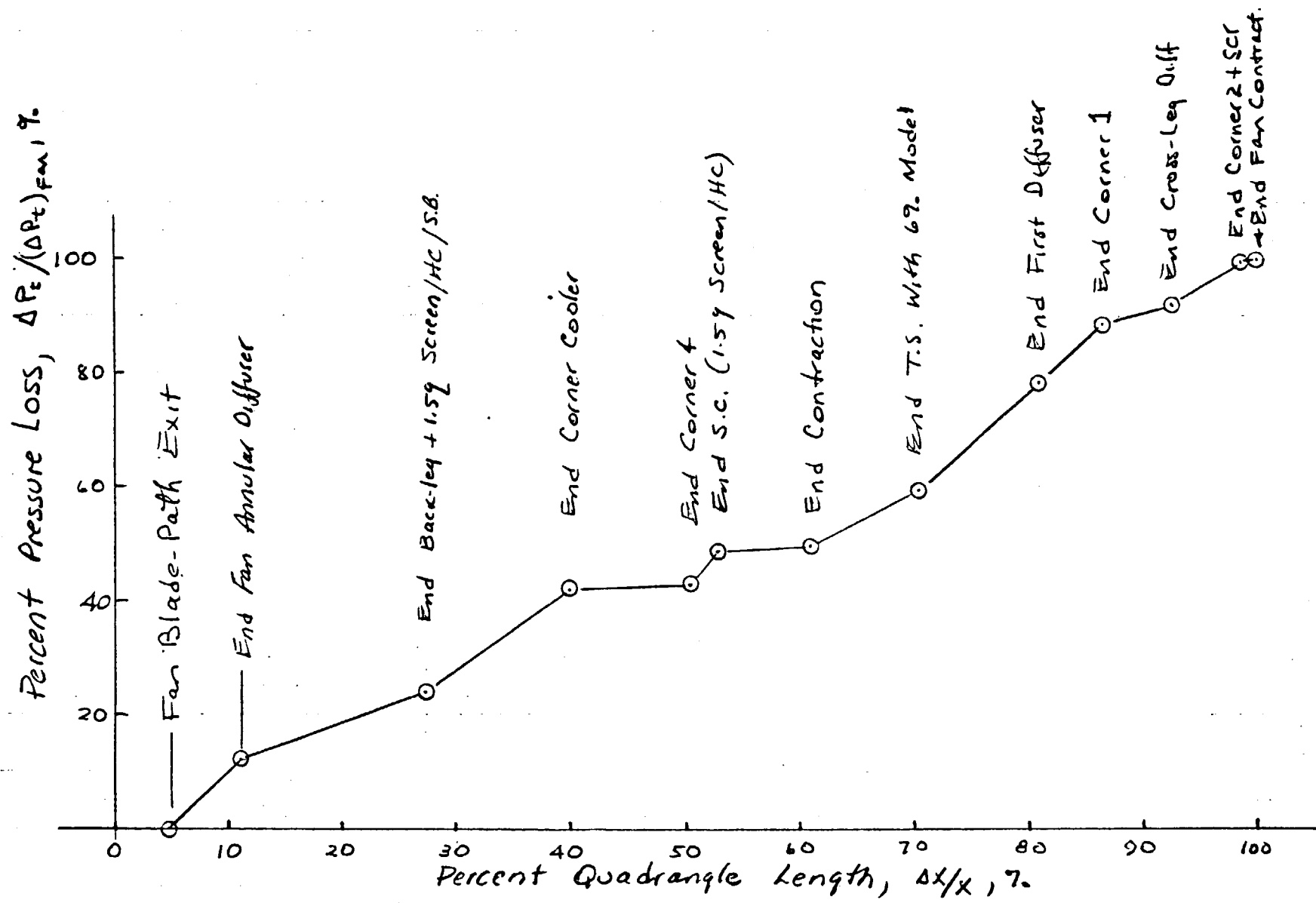
Max Alt. W/o PES

Adiabatic Efficiency Locus For Two Tunnels

(Main Drive Compressor Match)

MWG

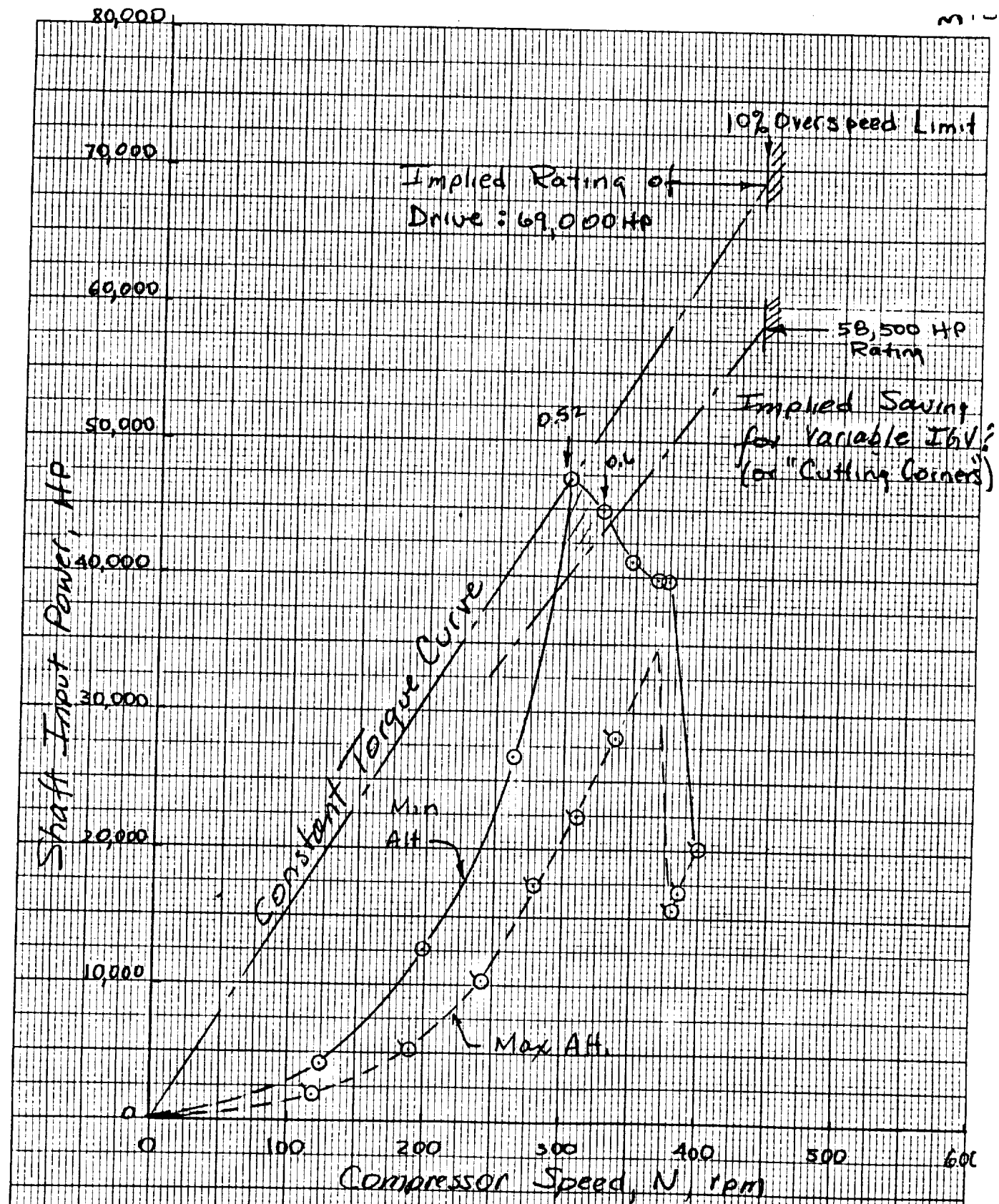
3/17/84



PES-Tunnel, $M=0.8$, $Z_0=32,000$ ft

Ducting Pressure Loss Build-up. With K3 Mod B Cooler
& Alternate ϕ Scoop With Tip

MWD
3/19/84



The Power-Speed Envelope PES-Tunnel
(Motor/Compressor Match)

MWD
 2/17/84

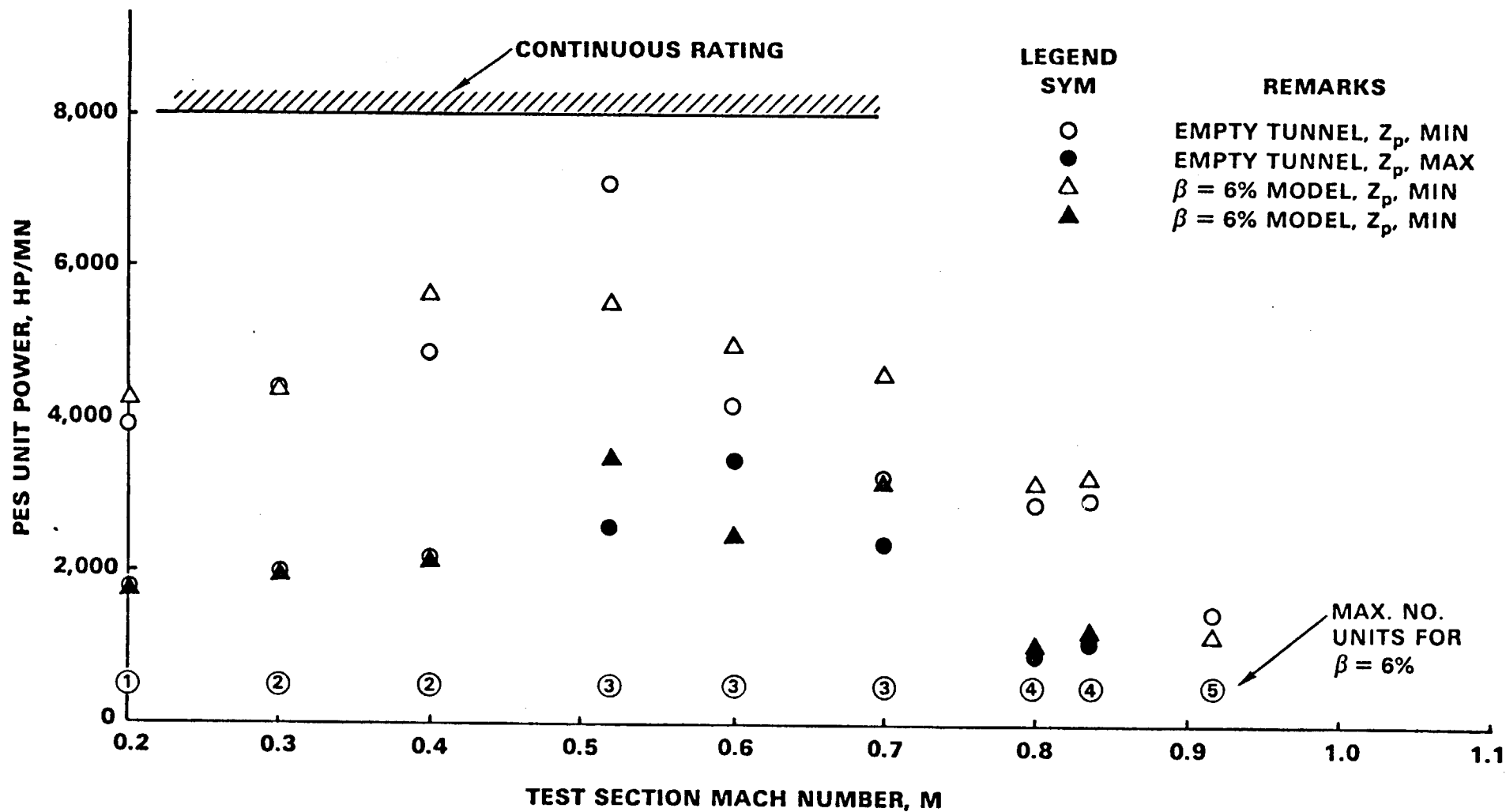


FIGURE 3. AWT-PES UNIT COMPRESSOR POWER FOR VARIOUS TEST CONDITIONS, DRY AIR W/O INLET HEATING

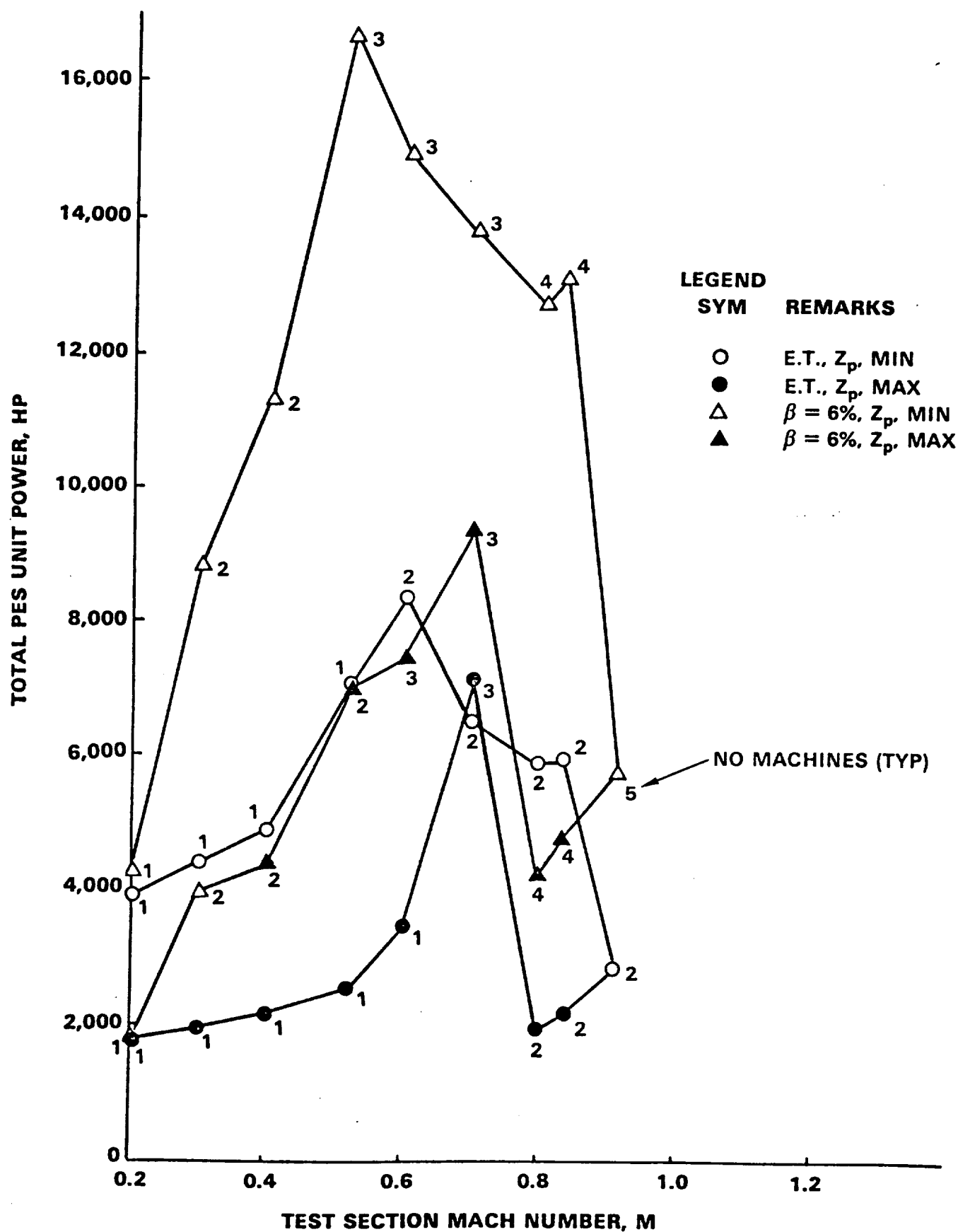
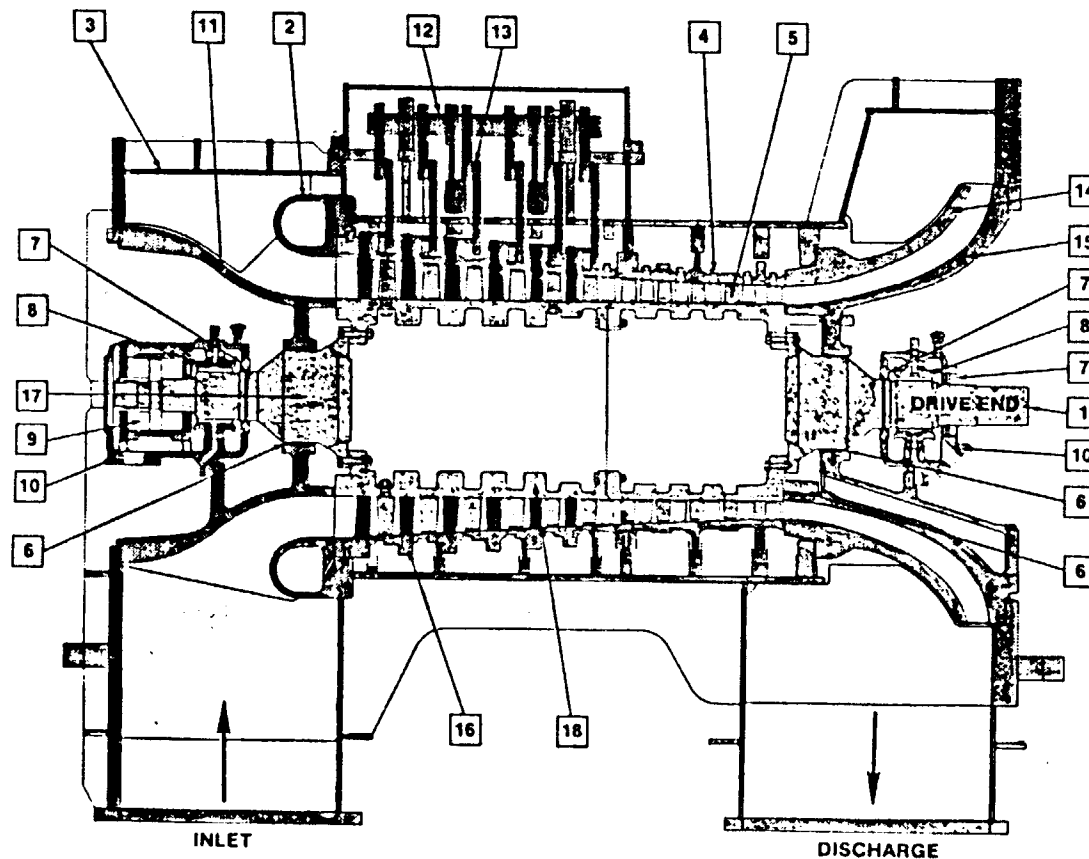


FIGURE 4. AWT-PES TOTAL COMPRESSOR POWER FOR VARIOUS TEST CONDITIONS, DRY AIR

construction features



- 1 Compressor Rotor Assembly
- 2 Fairing Ring
- 3 Casing
- 4 Stator Blading
- 5 Rotor Blading
- 6 Shaft Sealing "J-Strips"
- 7 Oil Baffle
- 8 Load Bearing
- 9 Thrust Bearing
- 10 Bearing Housing
- 11 Inlet End Bell
- 12 Operating Shaft
- 13 Movable Guide Vane Mech.
- 14 Diffuser
- 15 Disch. End Bell
- 16 Stator
- 17 Rotor Stub
- 18 Rotor Body

FIGURE 9. CONSTRUCTION FEATURES OF AN ALLIS-CHAMBERS 1410

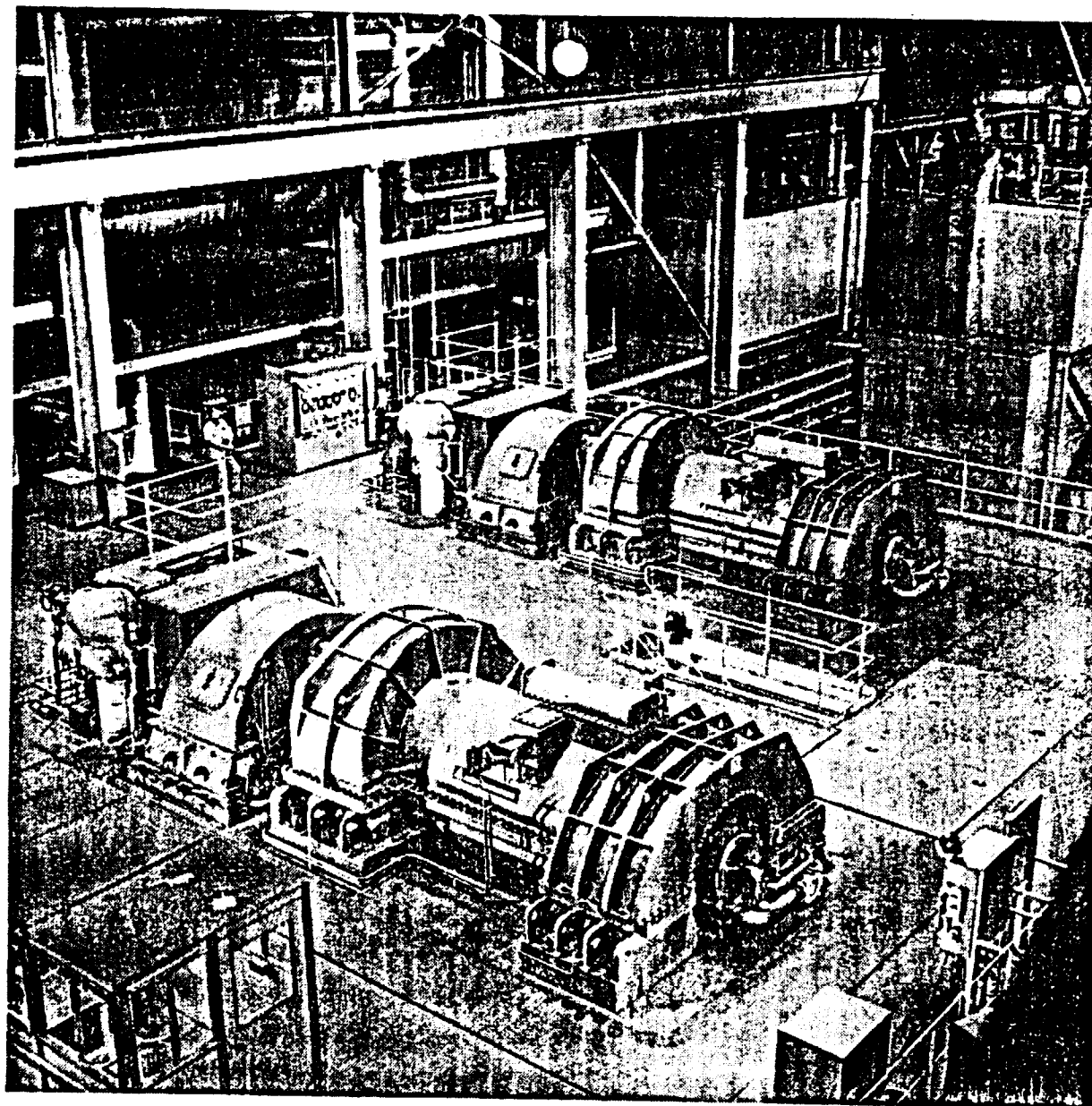
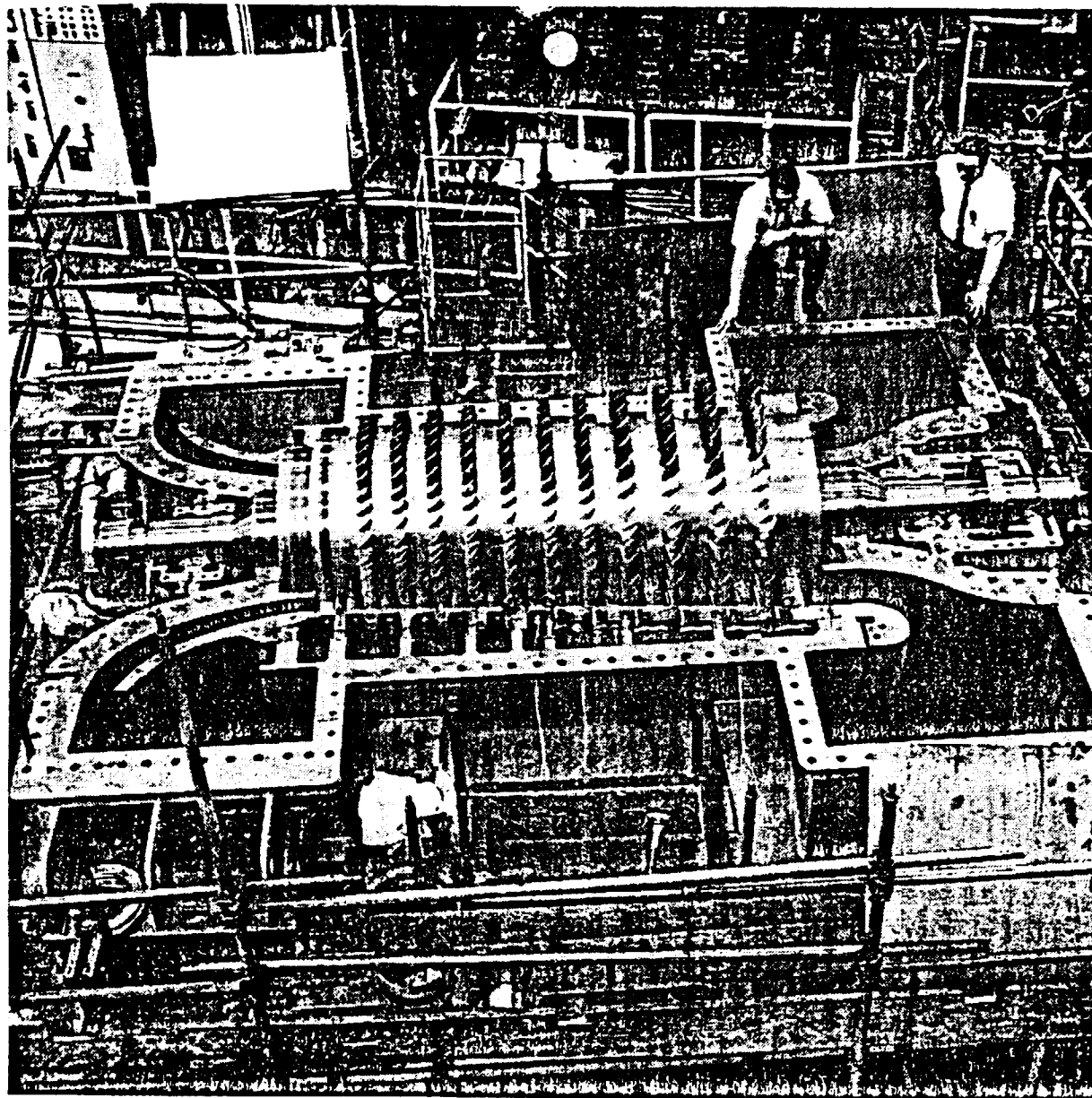


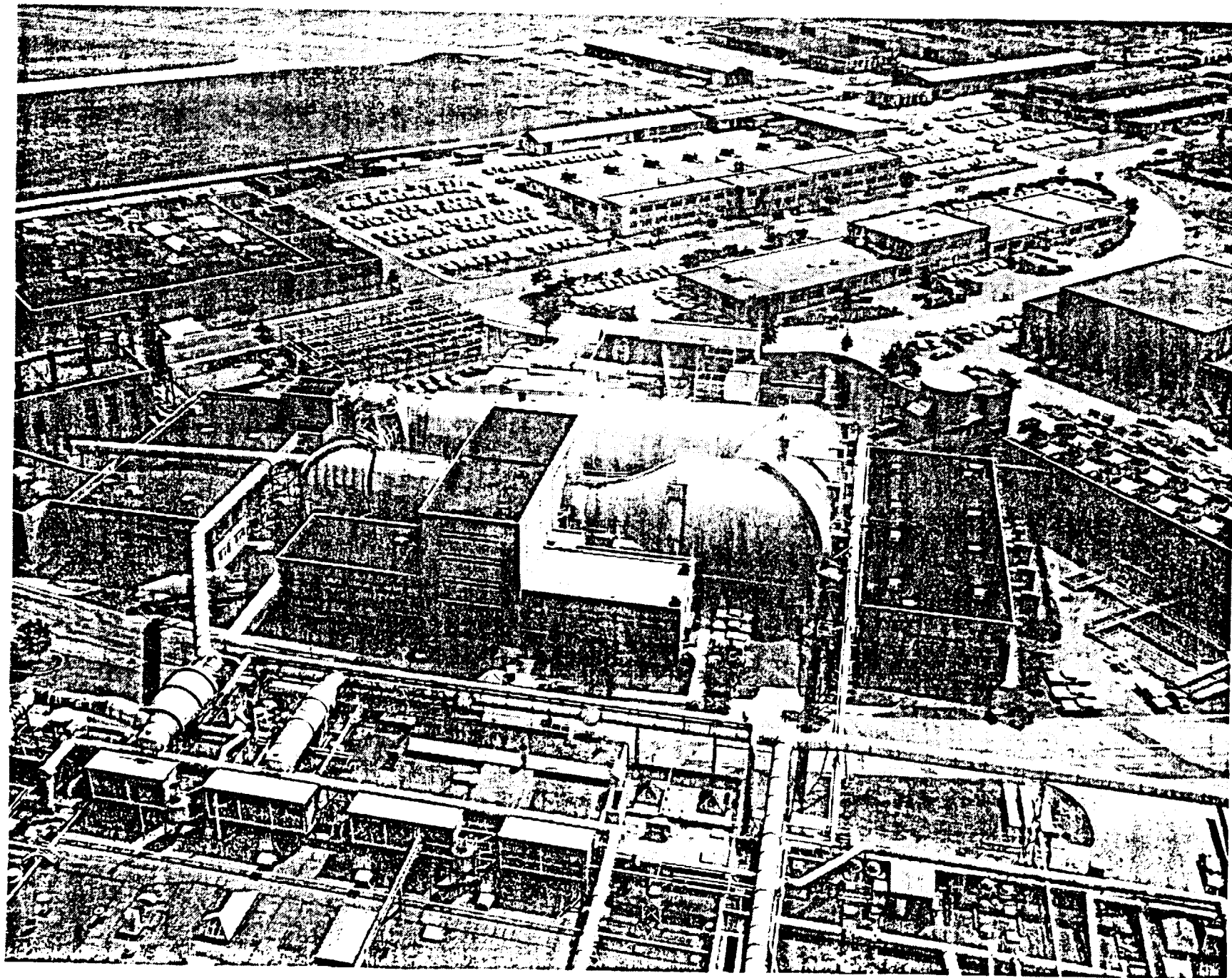
FIGURE 10. THE MAINTENANCE ADVANTAGE FOR A MEZZANINE LAYOUT OF AC 1410 MACHINES

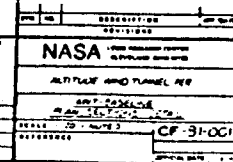


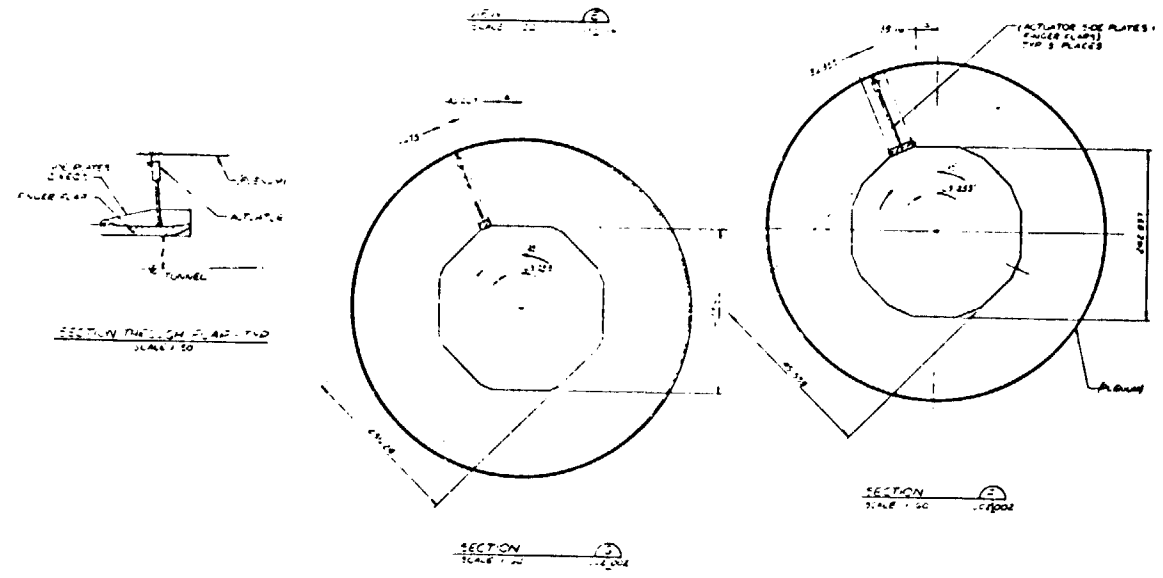
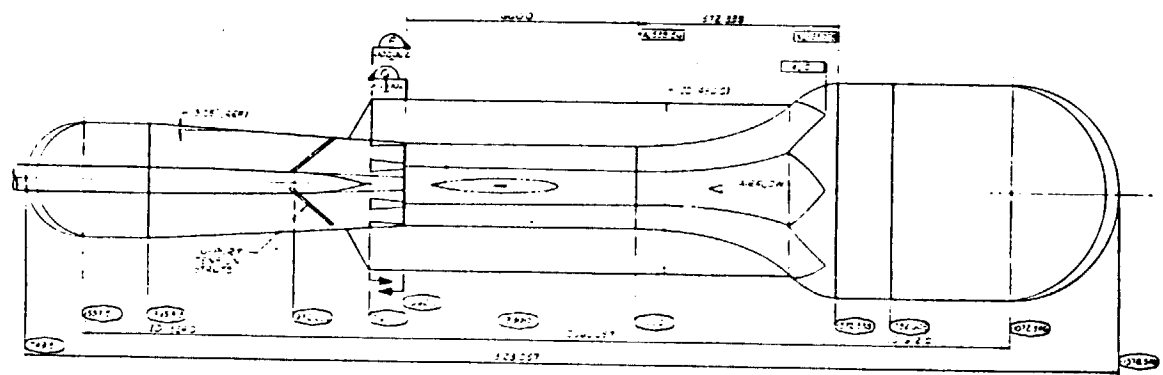
**FIGURE 11. THE MAINTENANCE ADVANTAGE FOR A CASING
WITH A HORIZONTAL SPLIT**

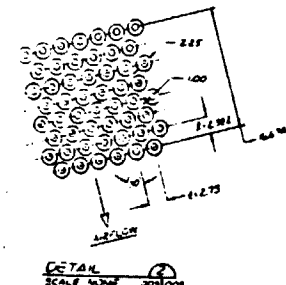
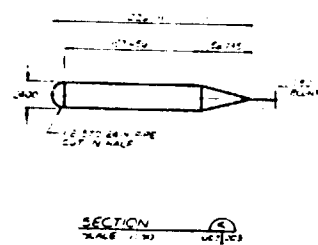
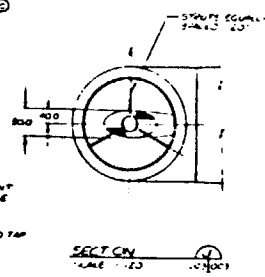
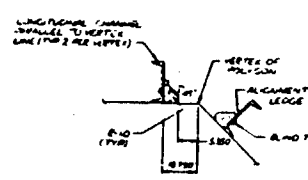
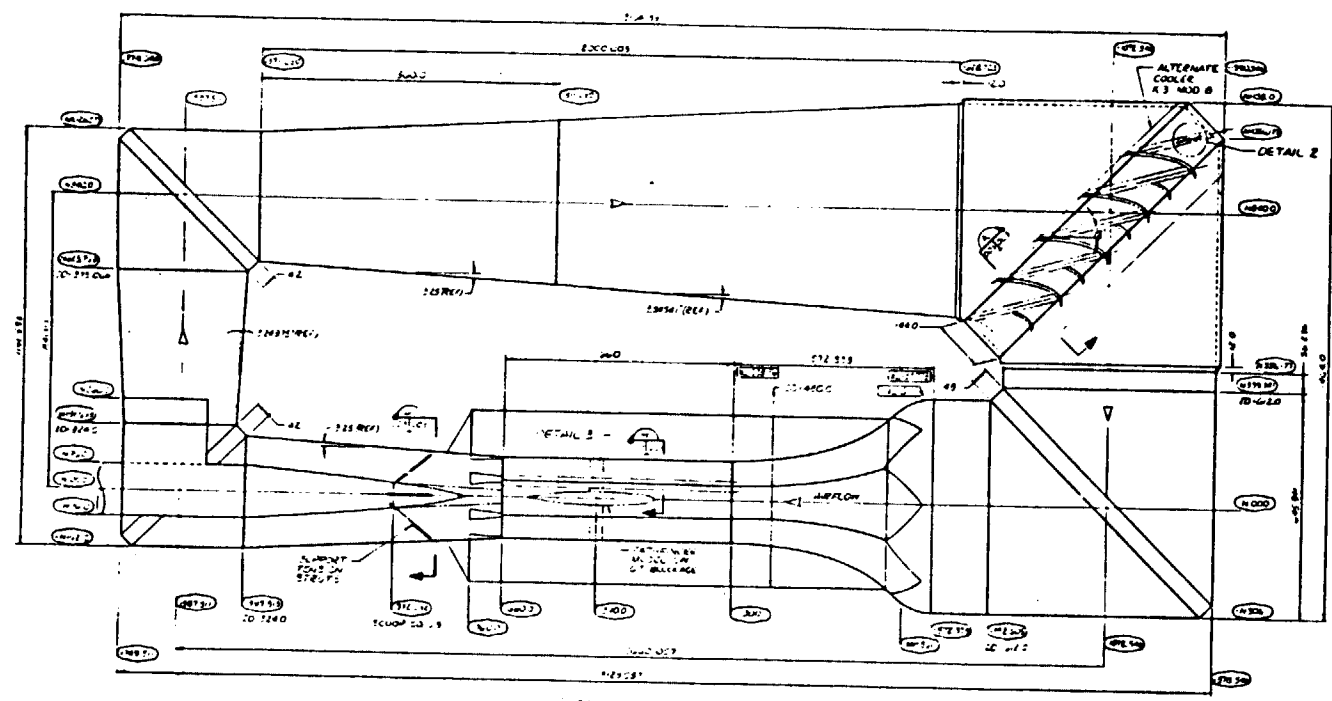
AIRLINE DEFINITION

- **BASELINE SCAVENGING SCOOP**
- **AEROFIN BASELINE MOD B COOLER**
- **K3, MOD B COOLER**





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TITLE: ALTITUDE AND TUNNEL TEST CELL DRAWING NO.: CF-10-100	
SCALE: 1/2" = 1'-0"	
DATE: 10-1-68	
DRAWN BY: J. L. HARRIS CHECKED BY: J. L. HARRIS APPROVED BY: J. L. HARRIS	

CONTRACTION GEOMETRY AND CODE ANALYSIS

TWO CONTRACTIONS WERE DESIGNED FOR AWT USING THE "MODIFIED SINE-LAW" APPROACH WITH BOUNDARY CONDITIONS, AS FOLLOWS:

- MOD A WAS CHOSEN TO BE 9.3786 FT. SHORTER THAN THE ORIGINAL AWT CONTRACTION
- MOD B WAS CHOSEN TO BE 15.0074 FT. SHORTER THAN THE ORIGINAL AWT CONTRACTION
- VNAP CODE RESULTS

MOD A. (9.3786 FT. SHORTER THAN ORIGINAL)

LAST HONEYCOMB/SCREEN, $\Delta X/D_{H,i}$						
	0.25		0.175		0.100	
	VEL fps.	% DEV.	VEL fps.	% DEV.	VEL fps.	% DEV.
WALL	793.97	+1.11	796.67	+1.24	796.37	+1.44
ξ	780.27	-0.53	781.61	-0.68	778.93	-0.78
AVG.	785.25		786.95		785.07	
TOTAL DEV		1.74		1.92		2.22

MOD B. (15.0074 FT. SHORTER THAN ORIGINAL)

LAST HONEYCOMB/SCREEN, $\Delta X/D_{H,i}$						
	0.25		0.174		0.100	
	VEL fps.	% DEV.	VEL fps.	% DEV.	VEL fps.	% DEV.
WALL	800.58	+1.34	788.10	+0.55	794.95	+1.25
ξ	791.94	+0.25	773.76	-1.18	779.03	-0.78
$\xi + 30$	779.94	-1.27				
AVG.	789.99		782.96		795.15	
TOTAL DEV		2.61		1.84		2.03

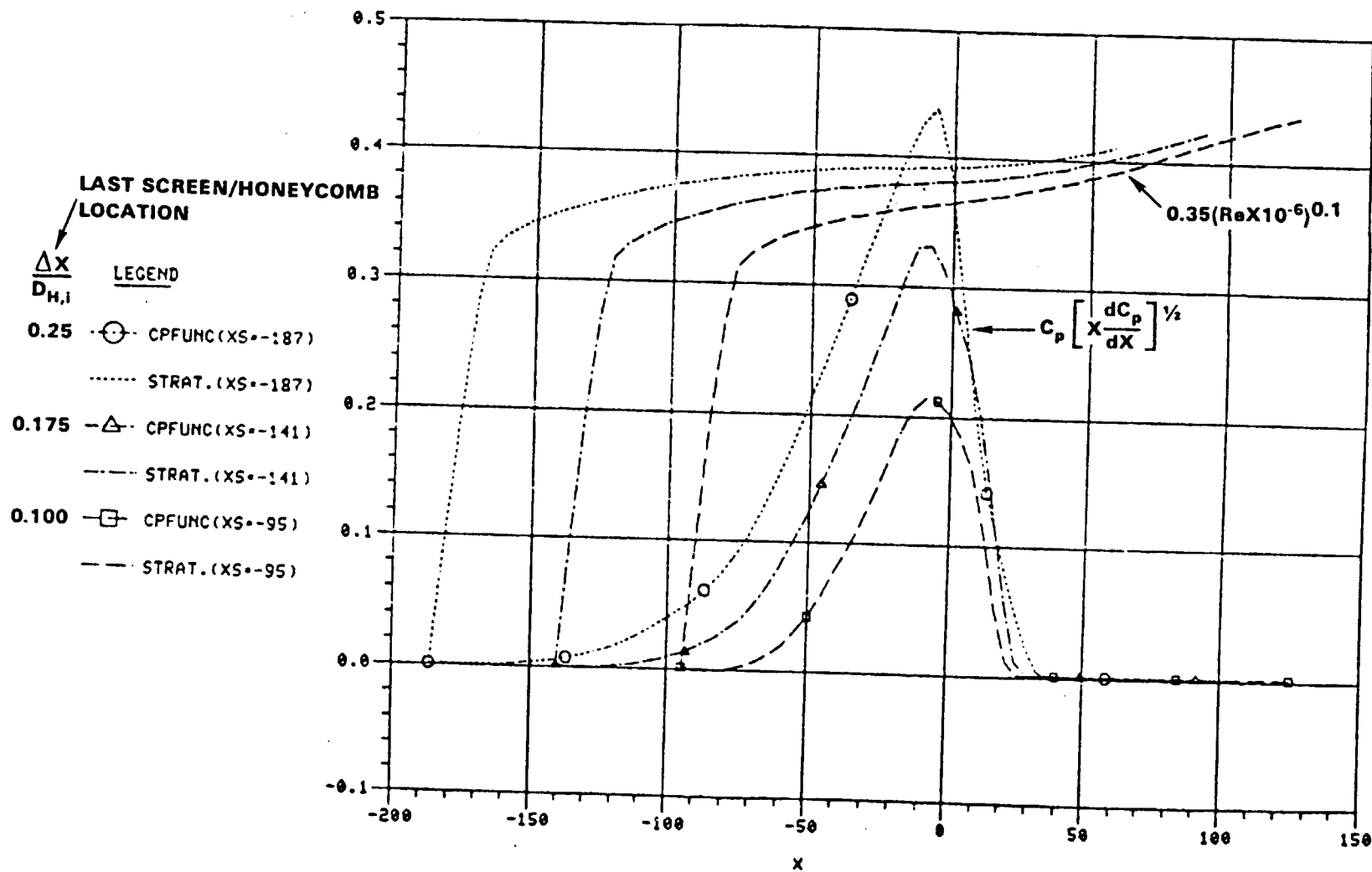


FIGURE 1. CHECK OF MOD A CONTRACTION FOR INLET FLOW SEPARATION (CONTRACTION 9.3786 FT. SHORTER)

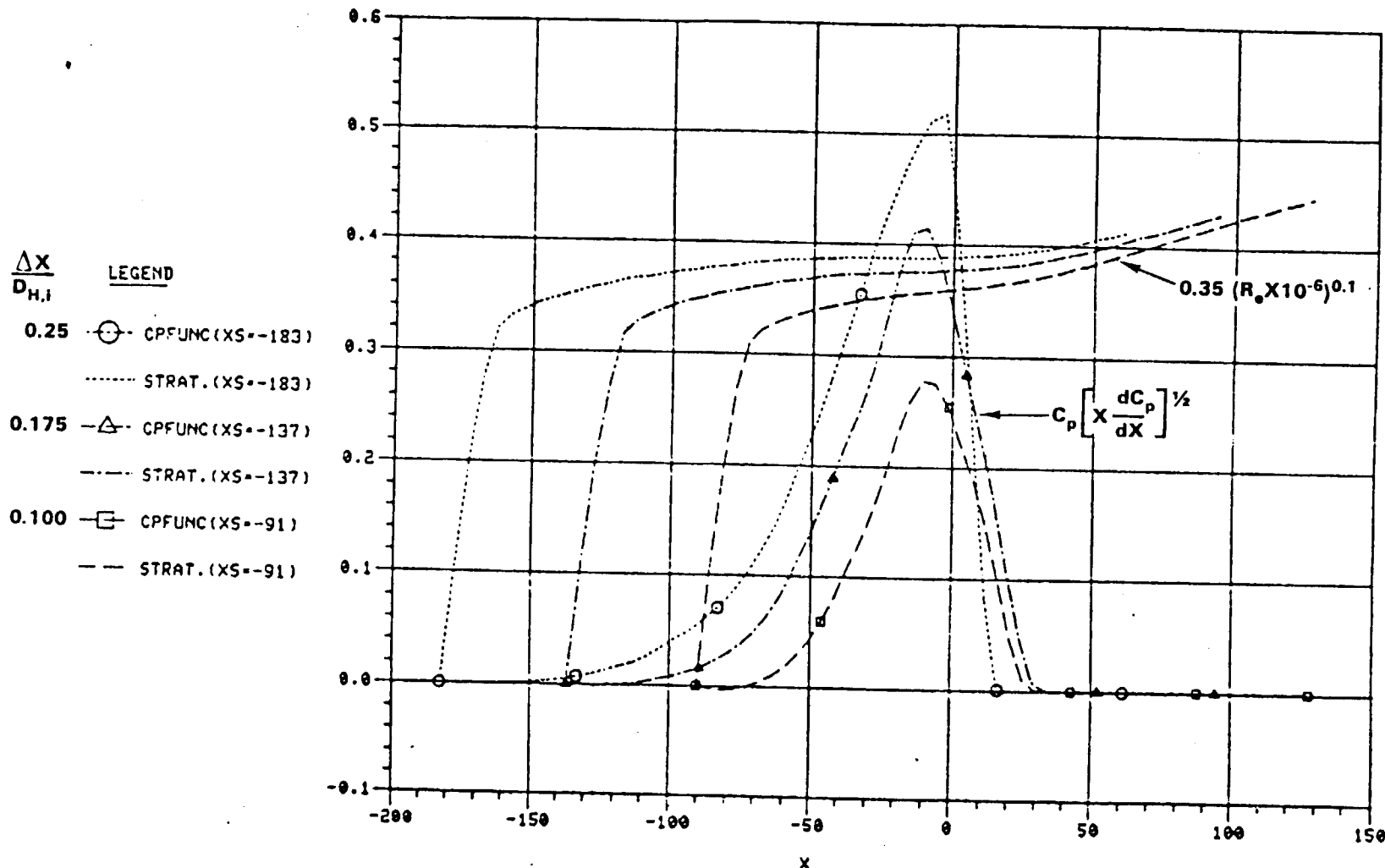


FIGURE 2. CHECK OF MOD B CONTRACTION FOR INLET FLOW SEPARATION (CONTRACTION 15.0074 FT. SHORTER)

CONTRACTION ANALYSIS (Concluded)

35
32

• INCOMPRESSIBLE FLOW CODE RESULTS

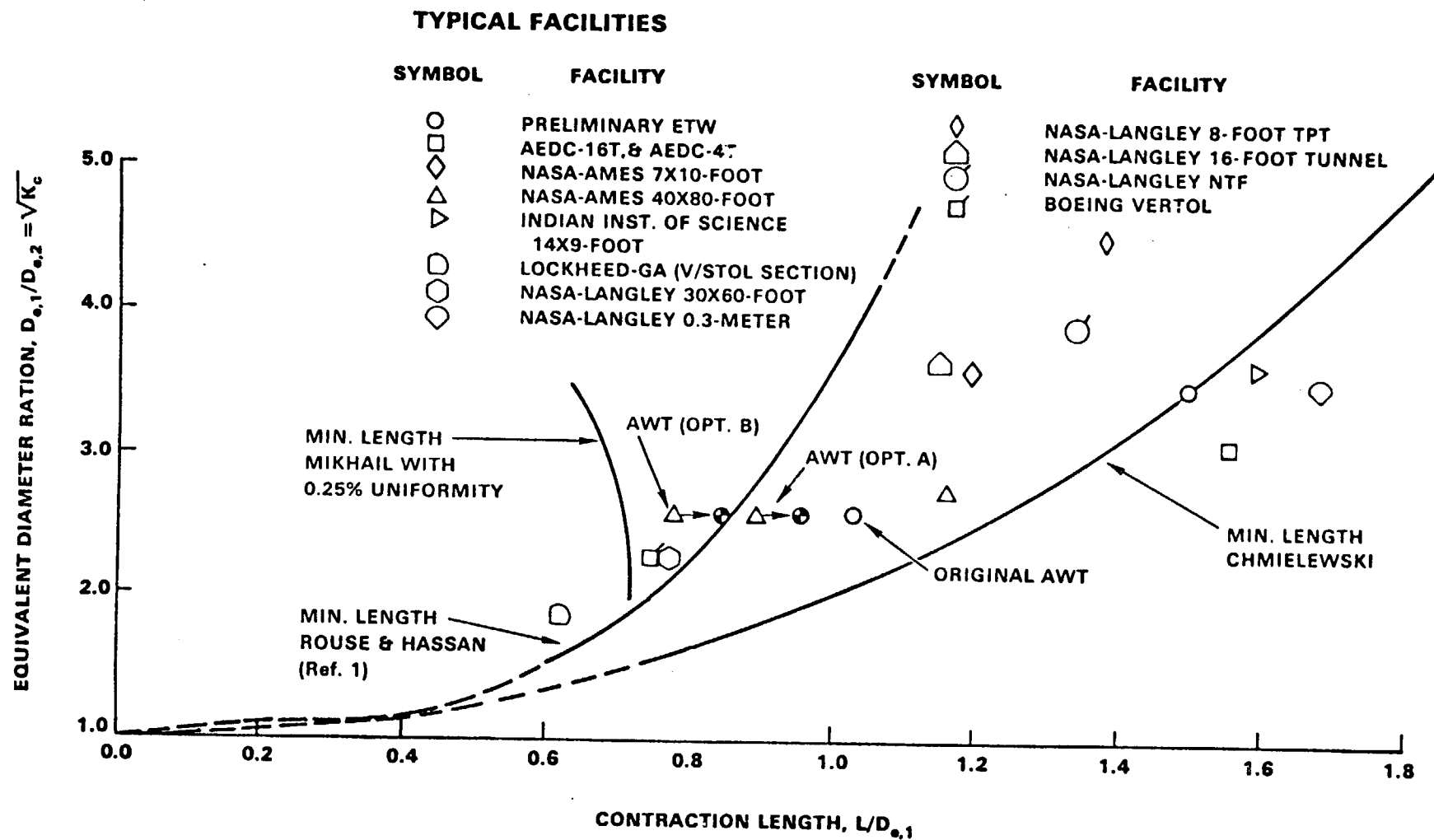
MOD A (9.3796 FT. SHORTER THAN ORIGINAL)

			WITH 10-FT. EXTENSION	
	V/V _{sc}	% DEV.	V/V _{sc}	% DEV.
WALL	6.556	+0.82	6.504	+0.02
ξ	6.437	-1.01	6.500	-0.05
AVG.	6.503		6.503	
	TOTAL DEV.	1.83		0.07

MOD B (15,0074 FT. SHORTER THAN ORIGINAL)

	V/V _{sc}	% DEV.	V/V _{sc}	% DEV.
WALL	6.580	1.18	6.504	+0.02
ξ	6.414	-1.37	6.499	-0.06
AVG.	6.503		6.503	
	TOTAL DEV.	2.55		0.08

• RECOMMEND MOD A WITH REMOVABLE SCREEN/HONEYCOMB SECTION



**FIGURE 3. SIMPLIFIED CONTRACTION LENGTH DESIGN CURVES
COMPARED TO SOME TYPICAL FACILITIES**

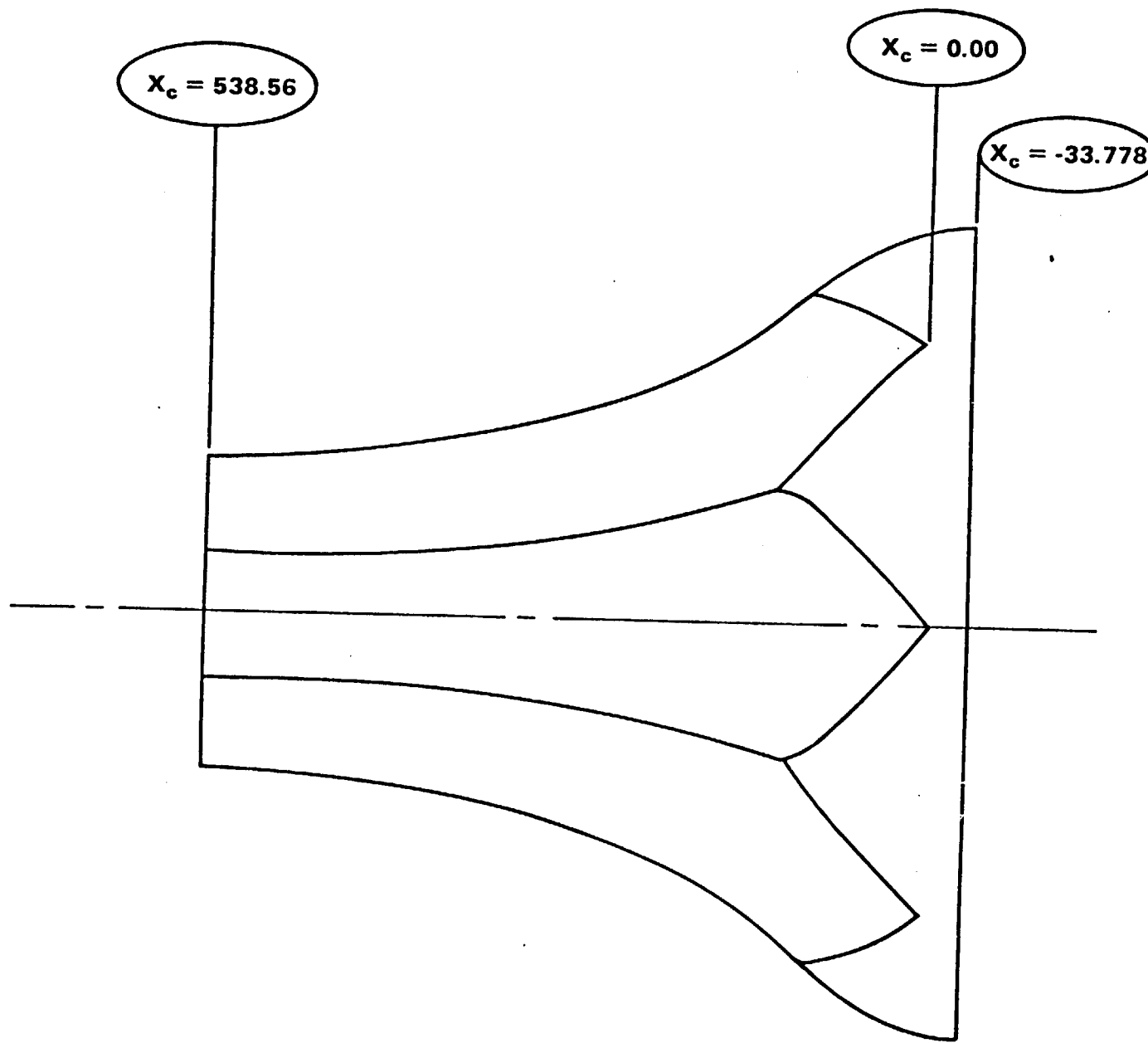
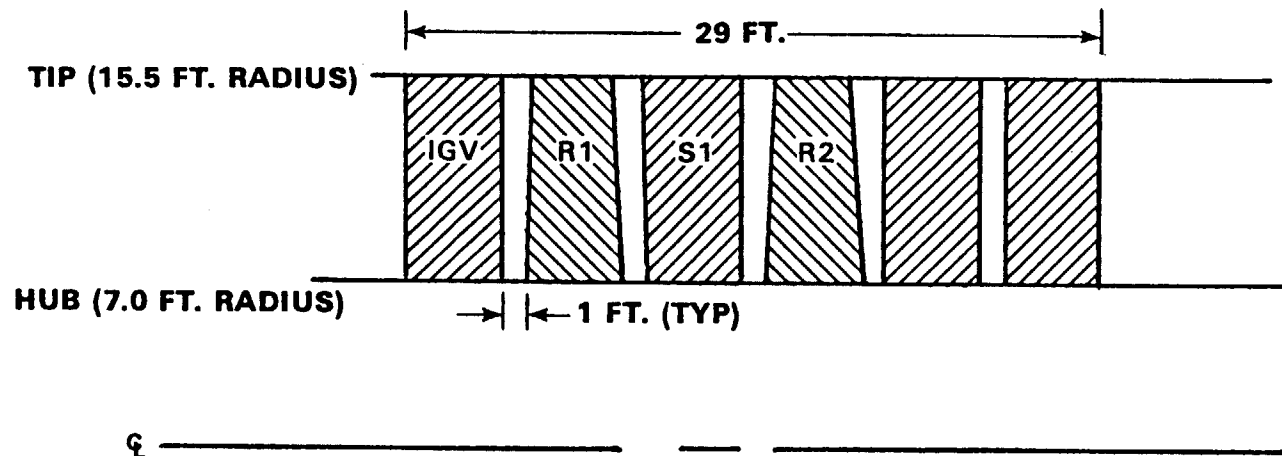


FIGURE 4. SCALE ELEVATION OF MOD A, AWT CONTRACTION

TABLE 1. AWT COMPRESSOR BLADE PATH BASELINE SPATIAL GEOMETRY

CHORD ROTOR IGV, STATORS, OGV

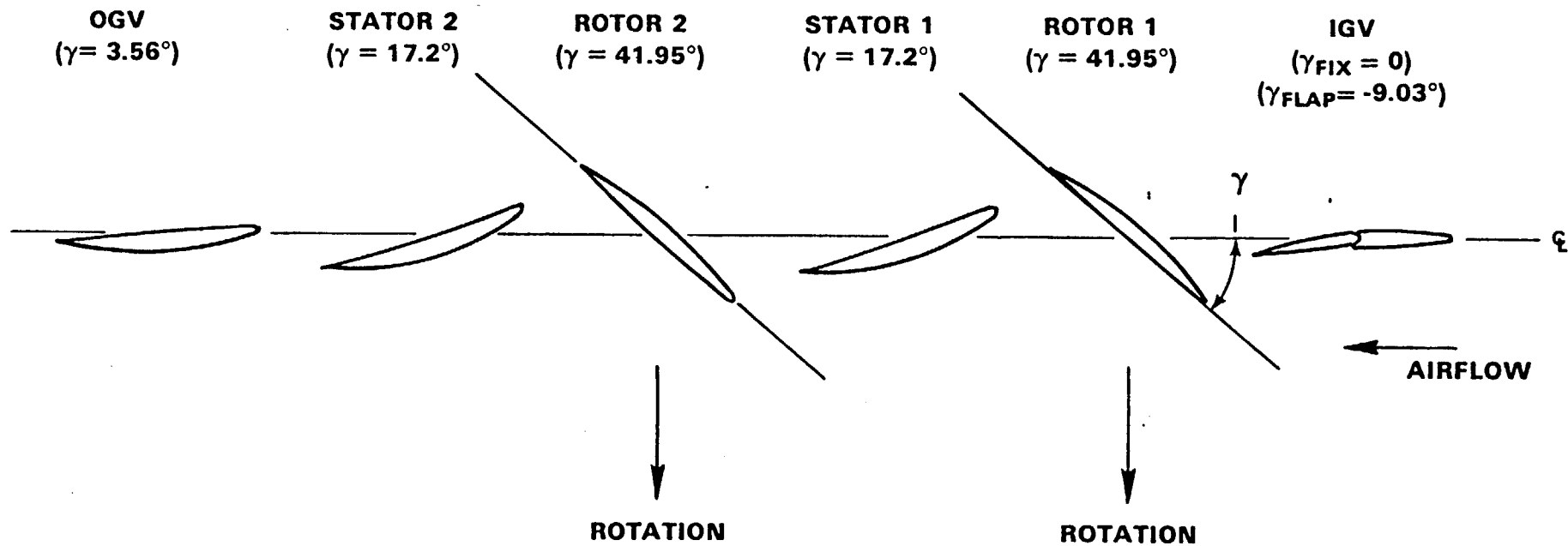
HUB 4.0 FT 4.0 FT
TIP 3.0 FT 4.0 FT

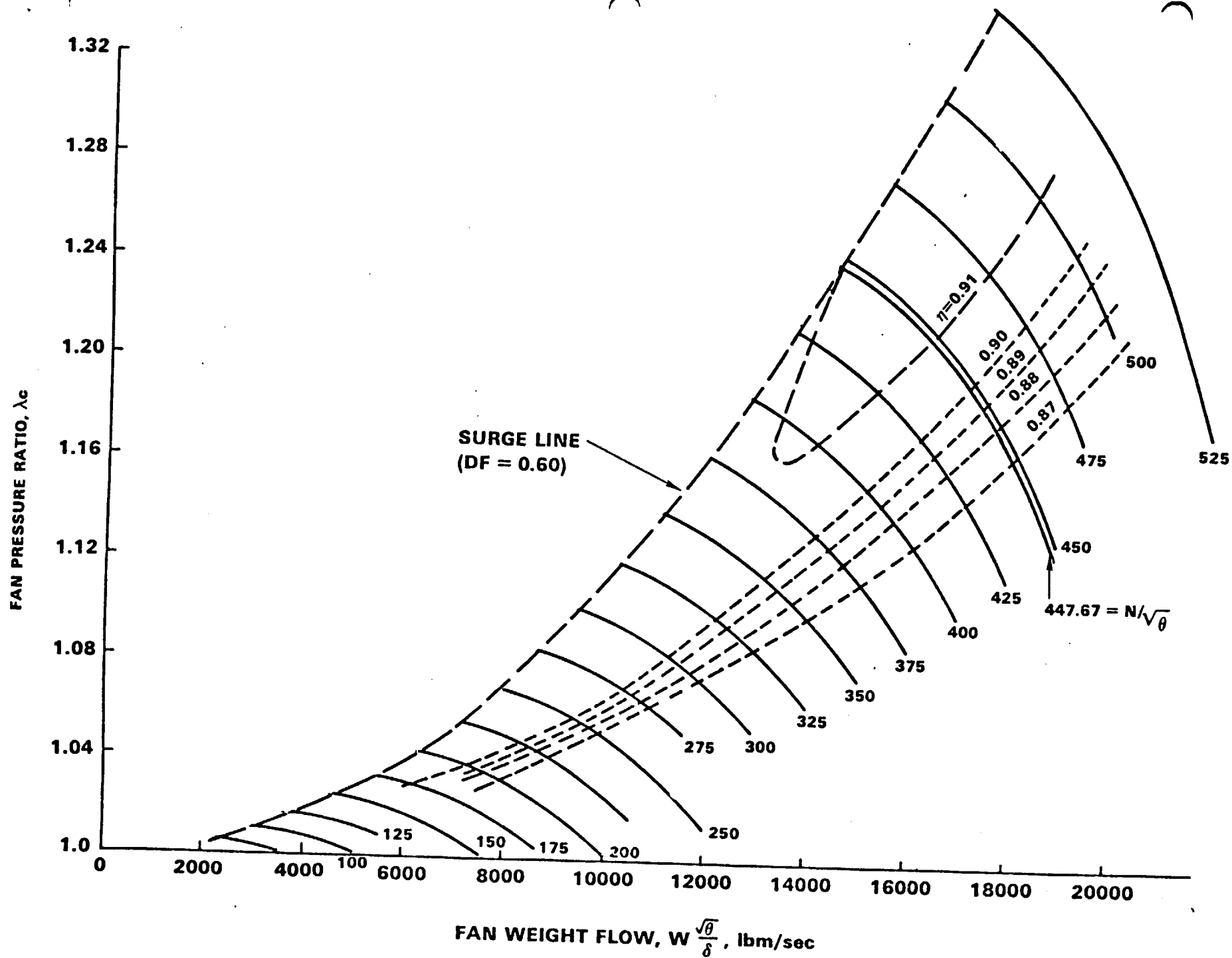


BLADE ROW	NO. OF BLADES	BLADE AIRFOIL SECIONS (CIRCULAR ARC MEANLINE)	
IGV	12	HUB: SvT MOD 65A010	TIP: SvT MOD 65A010
R1	17	SvT MOD 65A012	SvT MOD 65A006
S1	24	SvT MOD 65A010	SvT MOD 65A010
R1	17	SvT MOD 65A012	SvT MOD 65A006
S1	24	SvT MOD 65A010	SvT MOD 65A010
OGV	24	SvT MOD 65A010	SvT MOD 65A010

FAN BLADE PATH

RADIUS = 11.25 FT.
 γ = STAGGER ANGLE





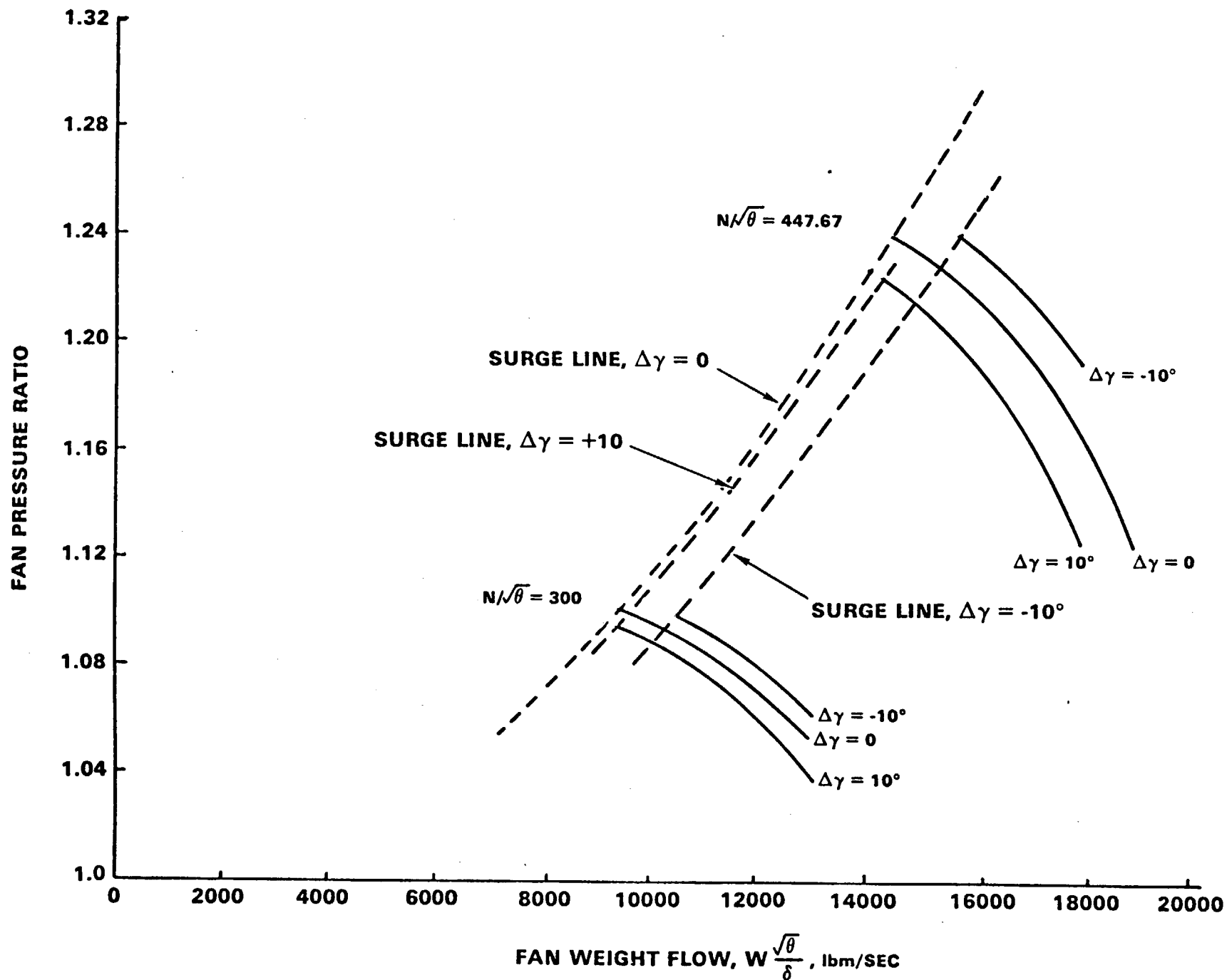


FIGURE 2. THE BASELINE AWT COMPRESSOR VARIABLE IGV CHARACTERISTICS

FAN ACOUSTIC PREDICTION METHOD

- BASED ON NASA TMX-71763
- CONSIDERS 1/3 OCTAVE BAND SPL
 - BROADBAND
 - DISCRETE TONE
 - COMBINATION TONE
- MODIFIED BOEING-AMES METHOD
- TONE CUT-OFF CRITERION INCLUDED
- CHARACTERISTIC 1/3 OCTAVE BAND SPL FOR SINGLE FAN STAGE:

$$L_c = F\left(\frac{\Delta T}{\Delta T_0}, \frac{\dot{m}}{\dot{m}_0}, M_{TR}, M_{TR_D}, RSS\right)$$

- SPL SPECTRUM:

$$SPL(f) = L_c + F\left(\frac{f}{f_b}\right)$$

- CORRECTION FOR INLET GUIDE VANES
- BASED ON DATA FOR 8 FULL-SCALE SINGLE-STAGE NASA-LEWIS FANS

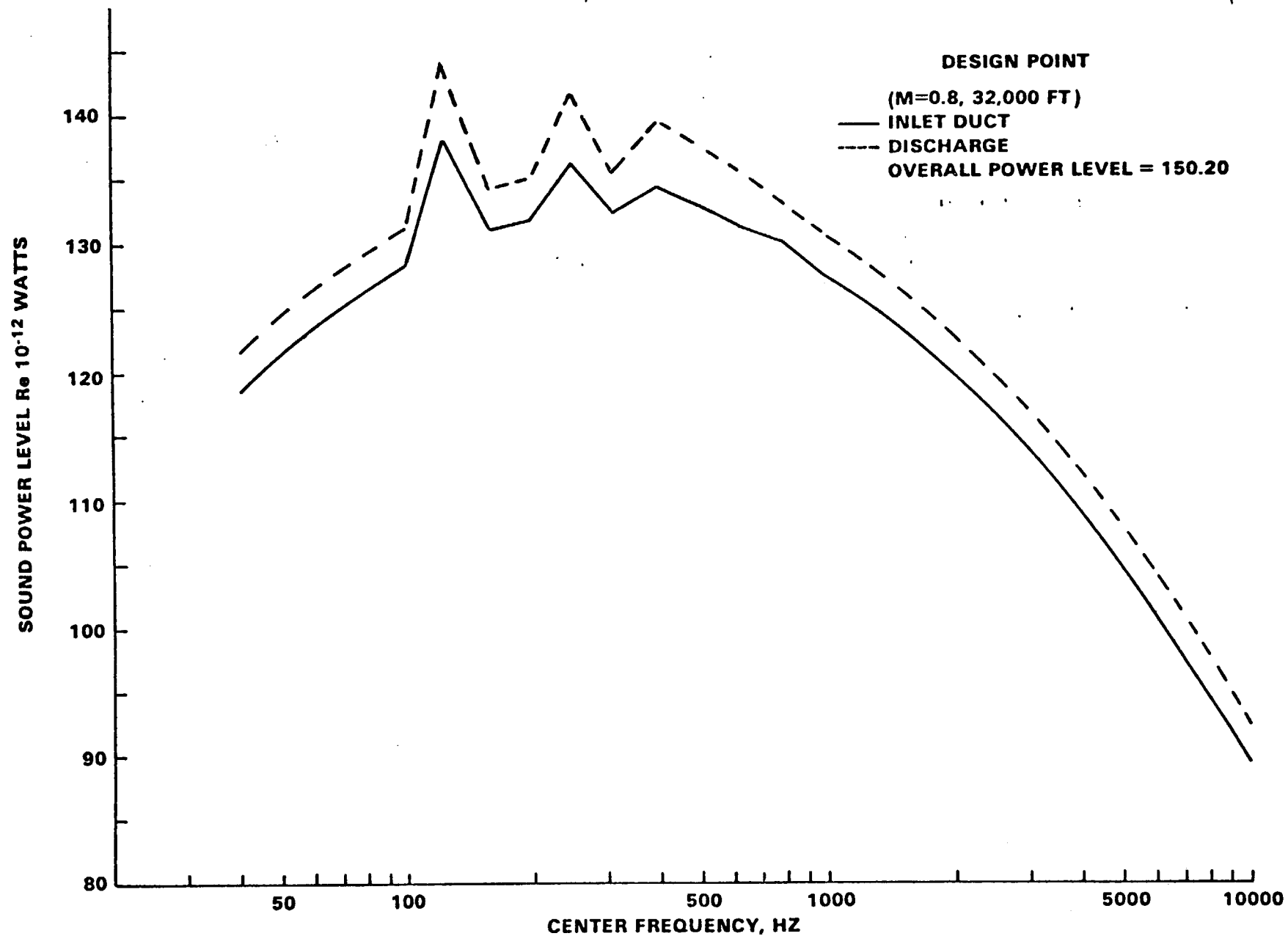


FIGURE 3. BASELINE AWT COMPRESSOR SOUND POWER SPECTRA
FOR THE DESIGN POINT

AWT FAN INLET DUCT

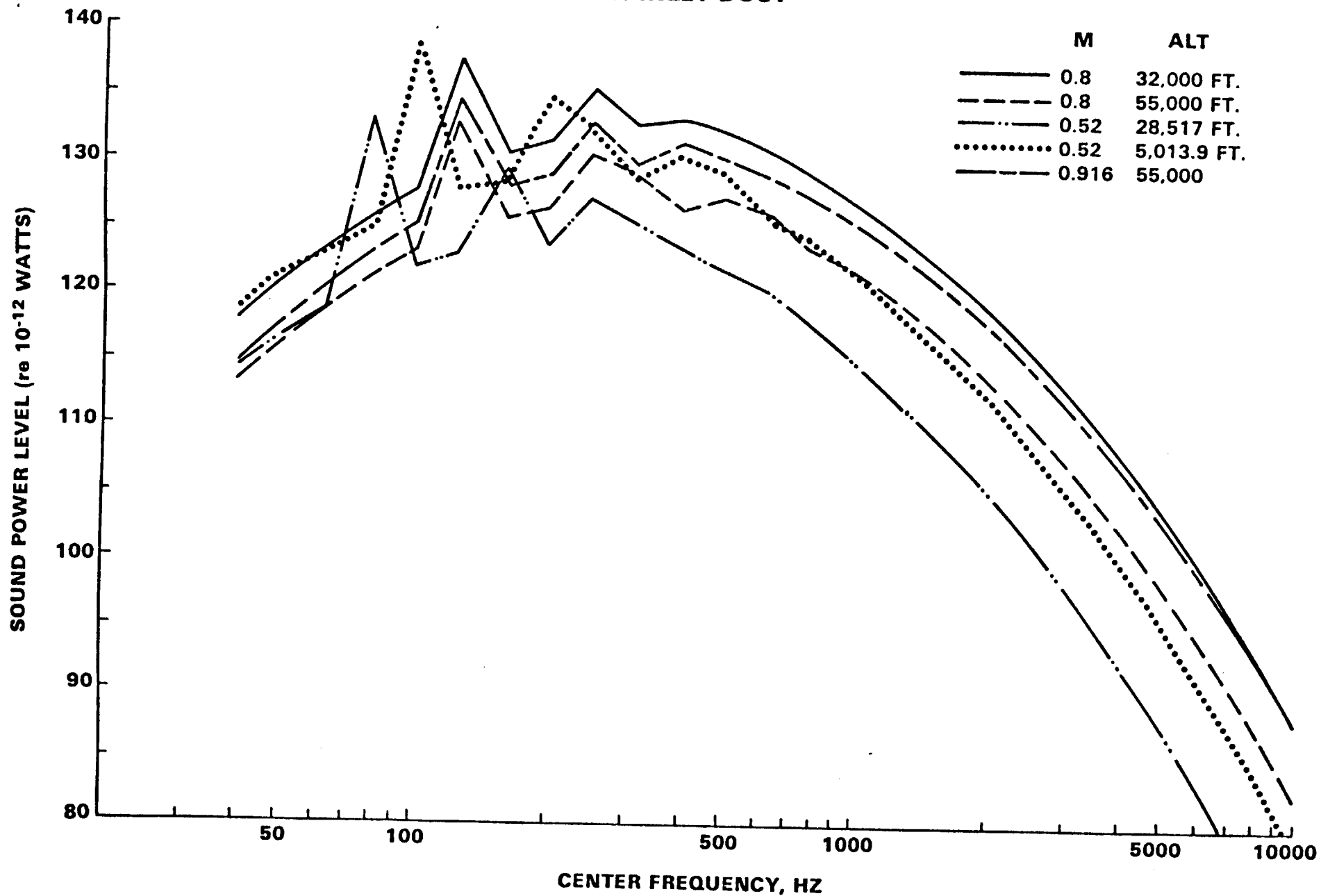


FIGURE 4. BASELINE AWT COMPRESSOR SOUND POWER SPECTRA AT THE MACHINE INLET FOR REPRESENTATIVE TEST CONDITIONS

AWT FAN DISCHARGE DUCT

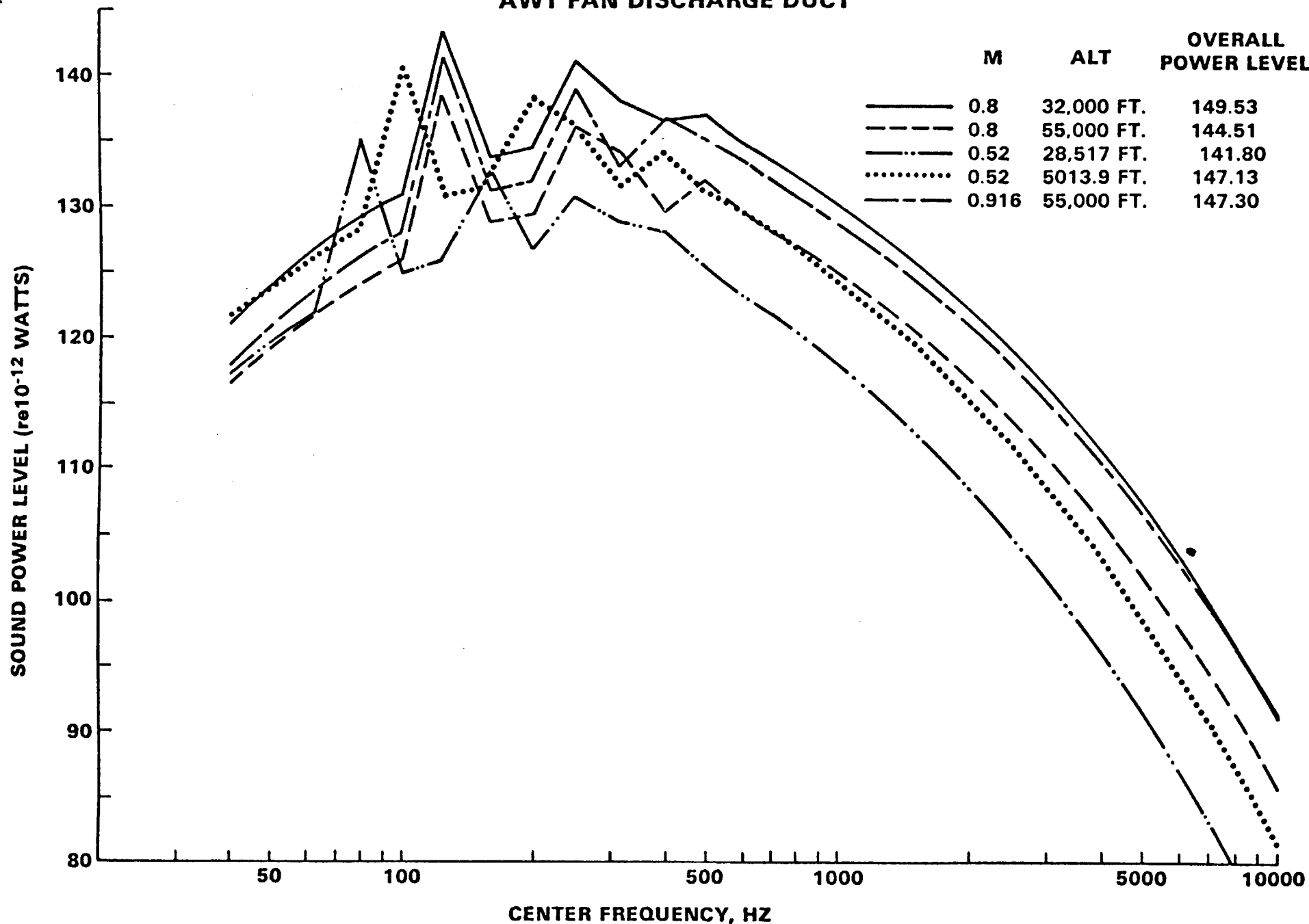
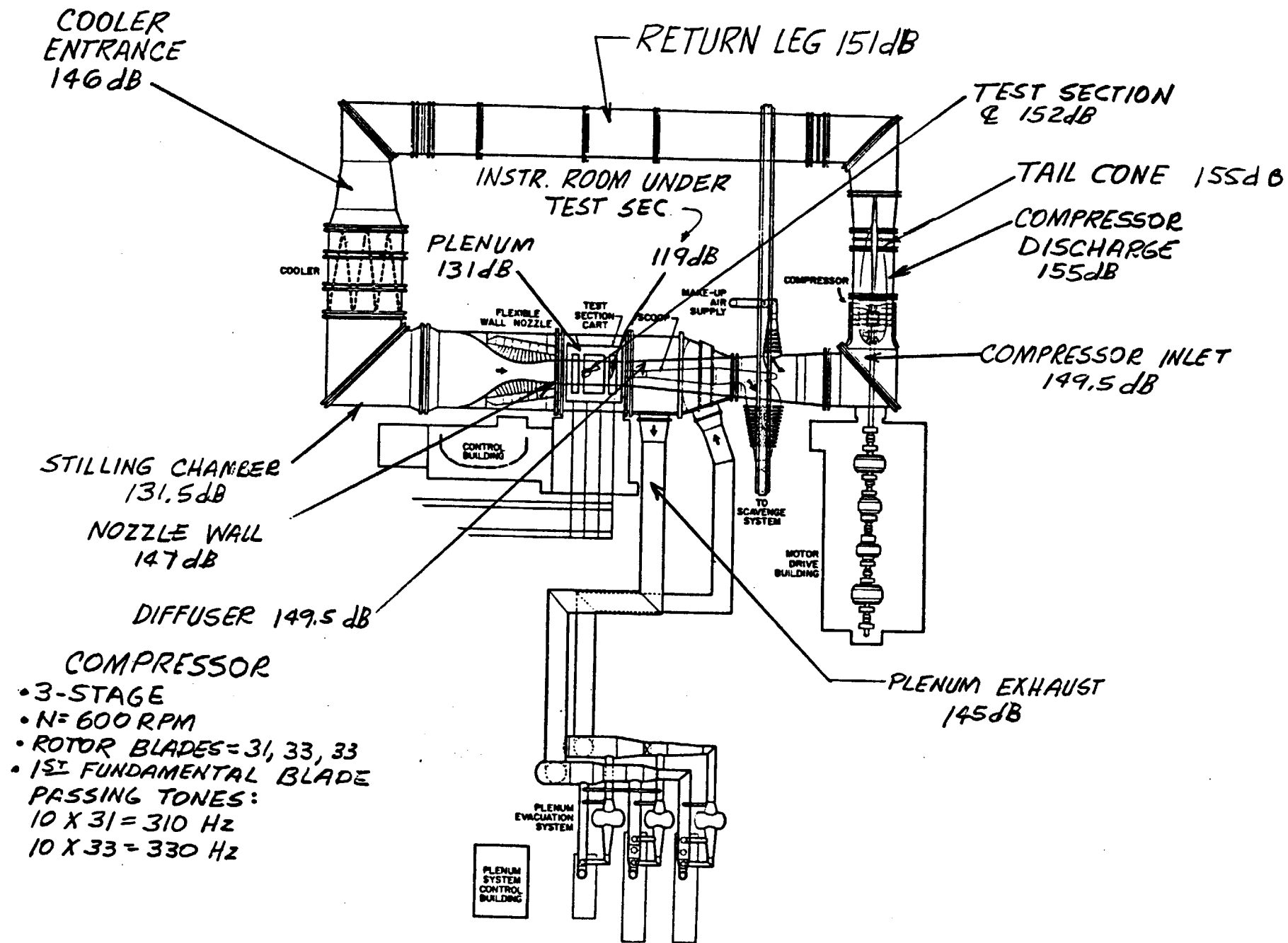


FIGURE 5. BASELINE AWT COMPRESSOR SOUND POWER SPECTRA AT THE MACHINE EXIT FOR REPRESENTATIVE TEST CONDITIONS



INTERNAL OVERALL SOUND POWER LEVELS IN TUNNEL 16T
 (M = 0.75 AND $P_t = 3100 \text{ PSFA}$)

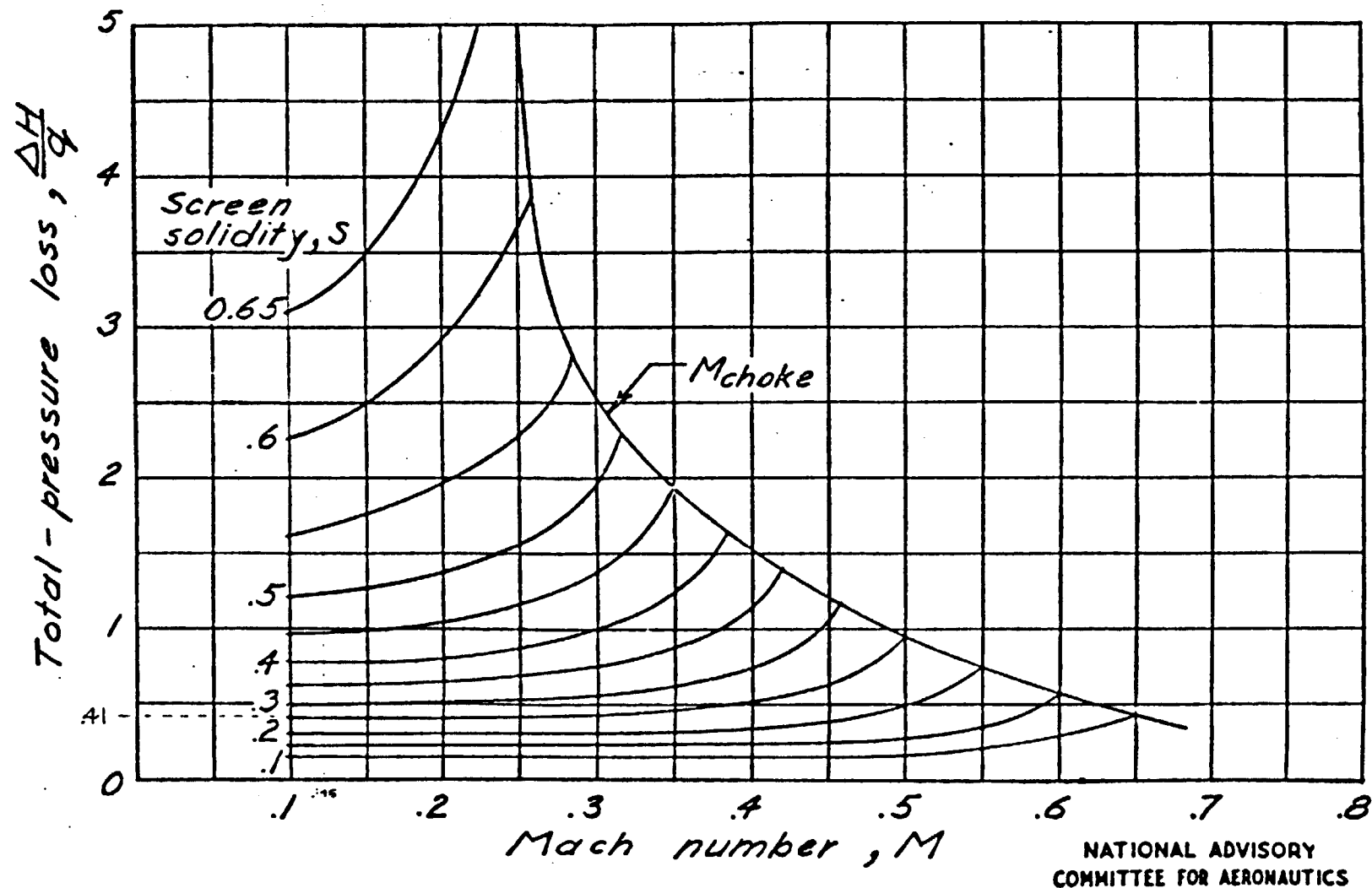


Figure 5. - Effect of compressibility on the total-pressure loss through screens of various solidities.

NACA CB No. L5F28

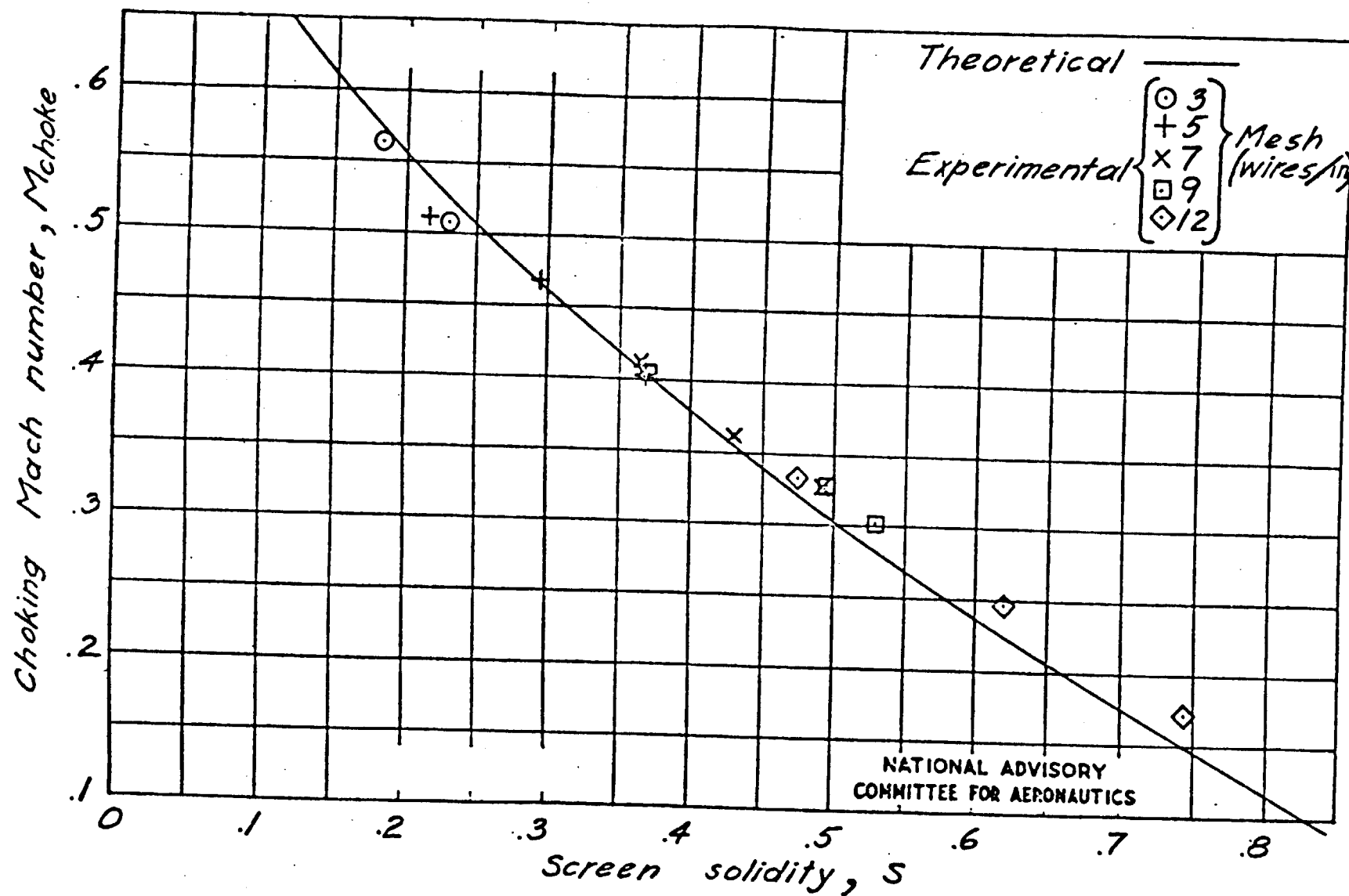


Figure 8. - Variation of choking Mach number with screen solidity.